High Performance Virtual Garment Simulation

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Abstract

For virtual characters the simulation of garments is a vital component towards realistic and believable scenarios that range from interactive virtual reality (virtual tailoring and cultural heritage) to realistic synthetic animation (CAD modeling and film production). This course addresses the key techniques involved in the latest state-of-theart in physically based cloth simulation.

Keywords: real-time simulation, cloth simulation, physically based modeling, garment animation, virtual characters.

1. Course Organizer

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2. Participants

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- Nadia Magnenat-Thalmann and Pascal Volino, University of Geneva, Switzerland.
- Markus Wacker, University of Tübingen and University of Applied Sciences Dresden, Germany.

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3. Synopsis

The course will present the following topics:

- Research fields in the core technologies for garment simulation: Prepositioning, mechanical models, numerical integration, collision detection and response.
- Application-specific techniques: Real-time virtual reality, garment design, fast prototyping.
- Demonstrations, projects, and case studies.

Necessary background

Fundamental knowledge in Computer Graphics and mathematics.

Potential Target Audience

Master and Ph.D. students, researchers, teachers.





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4. Course Outline

Introduction, overview and state of the art in cloth simulation, workflow in cloth simulation

A brief introduction to the topic of cloth simulation will reveal the relevant key technologies, the research challenges and the variety of application areas. Accompanied by a set of visual examples the audience will obtain a glance at the achievements in textile simulation as well as the demanding fields for future research. The necessary modules for a cloth simulation system will be highlighted, showing the whole pipeline from actual cloth materials and patterns to dressed virtual garments.

Material modeling and physical modeling

First, the mechanical properties of cloth and the corresponding measurements have to be identified, followed by their modeling on a discrete geometrical representation. We will introduce physical models for cloth ranging from simple mass-spring systems to continuum-based particle systems or finite element approaches. The accuracy of these models will be discussed relatively to their computational cost.



Prepositioning

To obtain an optimal initial position of the modeled garment patterns, which have to be sewed together, a prepositioning step is applied. Various semi-automatic or automatic methods will provide an interaction-free positioning of the planar garment parts around the person to be dressed.

Mathematical formulation and numerical treatment

The resulting equations then have to be mathematically analyzed and solved in time. We will describe several explicit and implicit time integration methods and compare them with respect to stability and performance. Discussion of the suitability of these methods to the various numerical requirements will be done according to the various simulation contexts (accurateness, interactivity, real-time).



Collision detection and response

During the simulation interaction of cloth with the environment and with itself is an important topic of its own. Efficient methods for addressing the difficult issue of fast collision detection are detailed and discussed. These include an overview of space subdivision algorithms (such as octrees) and object bounding volume hierarchies (such as Bounding Box-trees), along with specific details on efficient selfcollision detection. Moreover, the integration of geometrical collision information into mechanical contact forces and friction is addressed.

Applications

Particular attention is paid to the specific demands of the applications for cloth simulations ranging from high-quality CAD to interactive and real-time Virtual Reality. Selection criteria will be identified for the choice of the optimal cloth simulation methods for any given context. These include evaluating the accuracy and speed of various methods in order to render animations at frame-rates compatible with the intended applications. Requirements for robustness and stability will also be discussed. This session aims at illustrating the implementation of cloth simulation techniques in various contexts through a showcase of applications.



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One application is interactive garment design, where fast methods allow a garment designer to create and adjust garment models on virtual characters with help of interactive mechanical simulation. Discussion will show how interactivity is obtained through the use of dynamic meshes that allow mechanical computation to closely follow the current pattern shape and cloth properties of the edited garment.



Finally, a showcase of applications and ongoing projects will demonstrate the potentialities of the existing virtual garment simulation techniques in various fields involving virtual reality, such as virtual fashion design and virtual heritage.



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5. Participants

Wolfgang Straßer

(strasser@gris.uni-tuebingen.de) studied Electrical Engineering and Communications, and Computer Science at the Technical University of Berlin, where he received his Dipl.-Ing. (Masters in Engineering) degree in 1969. In 1974 he finished his Ph.D. work with his thesis work in the area of computer graphics hardware which describes for the first time the z-buffer algorithm. In 1978 he was appointed Professor of Computer Science at the Technical University Darmstadt. In 1986 he moved to University of Tübingen and founded the graphics research group. At present, Straßer is Professor of Computer Science and adjunct Professor of Mathematics at Tübingen. The graphics group in Tübingen consists of about 25 researchers working in the area of Graphics Systems Design, Graphics Hardware, Visualization, physical based modeling, Rendering and Geometric Modeling. The Lab is supported by grants from the German Science foundation, CEC and industry. In 1986, Straßer started the successful series of EG/Siggraph graphics hardware workshops. He has published numerous papers in scientific journals and conferences. He has given tutorials at EG, Siggraph, has chaired many conferences and workshops, and is a fellow of the EG Association. Straßer is a consultant to the government and industry. In 2000, the Technical University of Darmstadt awarded Professor Straßer with a honorary doctor degree for his outstanding contributions to the field of Computer Graphics.

Nadia Magnenat-Thalmann

(thalmann@miralab.unige.ch) has pioneered research into virtual humans over the last 25 years. She obtained several Bachelor's and Master's degrees in various disciplines (Psychology, Biology and Chemistry) and a Ph.D. in Quantum Physics from the University of Geneva. From 1977 to 1989, she was a Professor at the University of Montreal and led the research lab MIRALab in Canada. She moved to the University of Geneva in 1989, where she founded the Swiss MIRALab, an internationally interdisciplinary lab composed of about 30 researchers. She is author and coauthor of a very high number of research papers and books in the field of modeling virtual humans, interacting with them and in augmented life. She has received several scientific and artistic awards for her work, mainly on the Virtual Marylin and the film Rendez-Vous à Montreal. She has directed and produced several films and real-time mixed reality shows, among the latest are the Utopians (2001), Dreams of a Mannequin (2003) and the Augmented Life in Pompeii (2004). She is editor-in-chief of the Visual Computer Journal published by Springer Verlag and coeditor-in-chief of the Computer Animation & Virtual Worlds journal published by John Wiley.

Pascal Volino

(pascal@miralab.unige.ch) is a computer scientist, working at MIRAlab, University of Geneva. He is actually working on new models for cloth animation, involving versatile models for efficient simulations on situations involving high deformation, wrinkling and multilayer garments. The research is particularly focused on data structure, efficient collision detection, robust simulation and interactive cloth manipulation.

Markus Wacker

(wacker@informatik.htw-dresden.de) studied Mathematics and Physics at the University of Tübingen and the scuola normale superiore in Pisa, Italy. He received his Master degree in 1997. Afterwards he continued with his Ph.D. in Mathematics at the University of Tübingen in the field of functional analysis with research stays in Memphis, Tennessee, USA and Lecce, Italy. Since 2001 he is member of the graphics research group at the University of Tübingen and project leader of the national research project Virtual Try-On. He now has a professorship at the University of Applied Sciences Dresden for Computer Graphics. His main research areas are physically modeling of material parameters, finite element methods and design and tailoring applications in virtual reality.

Bernhard Thomaszewski

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Part 1: State-of-the-Art in Virtual Clothing

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Abstract

Virtual garment design and simulation involves a combination of a large range of techniques, involving mechanical simulation, collision detection, and user interface techniques for creating garments. Here, we perform an extensive review of the evolution of these techniques made in the last decade to bring virtual garments to the reach of computer applications not only aimed at graphics, but also at CAD techniques for the garment industry.

1. Introduction

The challenges of virtual garment simulation are numerous, and have attracted research efforts for more than a decade. First dedicated to the realistic simulation of the mechanical behavior of cloth, it soon evolved towards simulation of virtual garments on synthetic characters. While computer graphics gets the most obvious benefits from garment simulation on animated virtual characters, virtual prototyping of garment models is another major application field for the garment industry.

Virtual garment simulation is the result of a large combination of techniques that have also dramatically evolved during the last decade. Unlike the mechanical models used for existing mechanical engineering for simulating deformable structures, a lot of new challenges arise from the highly versatile nature of cloth. The central pillar of garment simulation obviously remains the development of efficient mechanical simulation models, which can accurately reproduce with the specific mechanical properties of cloth. However, cloth is by nature highly deformable and specific simulation problems arise from this fact. First, the mechanical representation should be accurate enough to deal with the nonlinearities and large deformations occurring at any place in the cloth, such as folds and wrinkles. Moreover, the garment cloth interacts strongly with the body that wears it, as well as with the other garments of the apparel. This requires advanced methods for efficiently detecting the geometrical contacts constraining the behavior of the cloth, and to integrate them in the mechanical model (collision detection and response). All these methods require advanced and complex computational methods where most important key issues remain computation speed and efficiency. For real-time applications however, only specific approximation and simplification methods allow the computation of garment animation, giving up some of the mechanical accuracy of the result in a result rather focused on visual realism.

Garment simulation, which started in the late eighties with very simple models such as Weil's approach [WEI 86], has taken much benefit from the increasing performance of computer hardware and tools as well as the development of specific simulation technologies which have nowadays lead to impressive applications not only in the field of simulation of virtual worlds, but also as design tools for the garment and fashion industry.

2. Initial Developments in Virtual Garment Simulation

In the field of computer graphics, the first applications for mechanical cloth simulation appeared in 1987 with the work of Terzopoulos et al [TER 87] [TER 88] in the form of a simulation system relying on the Lagrange equations of motion and elastic surface energy. Solutions were obtained through finite difference schemes on regular grids. This allowed simple scenes involving cloth to be simulated, such as the accurate simulation of a flag or the draping of a rectangular cloth. However, the first applications that really simulated garments started in 1990 (fig.1) with the considerations of many other technologies complementing cloth simulation [LAF 91] [CAR 92], such as body modelling and animation, and collision detection and response [YAN 93]. These applications innovated by providing the first virtual system allowing virtual garment patterns to be sewed together around a character.



Figure 1: "FlashBack": Early virtual garments used context-dependent simulation of simplified cloth models.

Since then, most developments were aimed at optimizing the accuracy and efficiency of the methods for simulating cloth accurately and efficiently, along the developments of actual applications and commercial products.

3. Mechanical Models

The accurate reproduction of the mechanical behaviour of cloth has always been a key issue for garment simulation. The mechanical behaviour of cloth is usually measured using standardized protocols, such as the Kawabata Evaluation System (KES), or the simpler FAST method, which are based on the experimental measurement of strain-stress curves for elongation, shearing and bending on normalized samples of fabric. Different representations of the cloth surface mechanics then allow the virtual reproduction of the behaviour of cloth.

Well known in mechanical engineering, the Finite Element method considers the cloth surface as being discretized in interpolation patches for a given order (bilinear, trilinear, quadrilinear), and an associated set of parameters (degrees of freedom) that give the actual shape to the interpolation surface over the element. From the mechanical properties of the material, the mechanical energy is computed from the deformation of the surface for given values of the interpolation parameters. An equation system based on the energy variation is then constructed with these degrees of freedom. Surface continuity between adjacent elements imposes additional constraint relationships. A large sparse linear system is built by assembling successively the contributions of all the elements of the surface, and then solved using optimized iterative techniques, such as the conjugate gradient method.

Finite elements have only had a marginal role in cloth simulation. The main attempts are described in [COL 91], [GAN 95], [EIS 96]. Most implementations focus on the accurate reproduction of mechanical properties of fabrics, but restrict the application field to the simulation of simple garment samples under elementary mechanical contexts, mostly because of the huge computational requirements of these models. Furthermore, accurate modelling of highly variable constraints (large nonlinear deformations, highly variable collisions) is difficult to integrate into the formalism of finite elements, and this sharply reduces the ability of the model to cope with the very complicated geometrical contexts which can arise in real-world garment simulation on virtual characters.

An easier and more pragmatic way to perform cloth simulation is the use of particle systems. Particle systems consider the cloth to be represented only by the set of vertices that constitute the polygonal mesh of the surface. These particles are moved through the action of forces that represent the mechanical behaviour of the cloth, which are computed from the geometric relationships between the particles that measure the deformation of the virtual cloth. Among the different variations of particle systems, the spring-mass scheme is the simplest and most widely used (fig.2). It considers the distance between neighbouring particle pairs as the only deformation measurement and interaction source representing the internal elasticity of the cloth.

Particle systems are among the simplest and most efficient ways to define rough models that compute highly deformable mechanical systems such as cloth with computation times small enough to integrate them into systems for simulating complete garments on virtual bodies. Among the main contributions on particle system models, early works considered simple viscoelastic models on regular grids with applications for draping problems with simple numerical integration schemes [SAK 91]. Accurate models started with Breen et al [BRE 94] on modelling the microstructure of cloth using parameters derived from KES behaviour curves and integration based on energy

minimization. However, such accurate models required a lot of computation for solving problems that were restricted to draping. On the other hand, more recent models trade accuracy for speed, such as the grid model detailed by Provot et al [PRO 95] which additionally includes geometric constraints for limiting large deformation of cloth. Additional contributions from Eberhardt et al [EBE 96] with the simulation of KES parameters and comparison of the efficiency of several integration methods. Advanced surface representations were used in [DER 98], where the simulation model and collision detection takes advantage of the hierarchical structure of subdivision surfaces. Modelling animated garments on virtual characters is the specific aim of the work described by Volino et al [VOL 95] [VOL 97], which investigate improved spring-mass representations for better accuracy of surface elasticity modelling on irregular meshes.



Figure 2: Particle systems model cloth surfaces as point masses interacting with their neighbours using forces computed from their relative positions.

While various models can be used to compute the force applied on each particle given their position and speed, these forces have then to be integrated along time to obtain the position and speed of the particle for the following timesteps using methods related to the integration of ordinary differential equation systems. Most recent however focus on improvements of the numerical integration methods in order to improve efficiency of the simulation.

Explicit integration methods are the simplest methods available for solving first-order ordinary differential systems. They consider the prediction of the future system state directly from the value of the derivatives. The best known techniques are the Runge-Kutta methods. Among them, the fast but unstable and inaccurate first-order Euler method, used in many early implementations, considers the future state as a direct extrapolation from the current state and the derivative. Higher order and more accurate methods also exist, such as the second-order Midpoint method, used for instance in early models by Volino et al [VOL 95], and the very accurate fourth-order Runge-Kutta method, used for instance by Eberhardt et al [EBE 96].

Beside considerations for accuracy, stability and robustness are other key factors to consider. For most situations encountered in cloth simulation, the numerical stiffness of the equations (stiff elastic forces, small surface elements) require the simulation time-steps to be small enough to ensure the stability of the system, and this limits the computation speed much more than accuracy considerations. Adequate time-step control is therefore essential for an optimal simulation. A common solution is to use the fifth-order Runge-Kutta algorithm detailed in [PRE 92] which embeds integration error evaluation used for tuning the time-step adaptively [VOL 97].

In order to circumvent the problem of instability, implicit numerical methods are being used. For cloth simulation, this was first outlined by Baraff et al [BAR 98]. The most basic implementation of implicit method is the Euler step, which considers finding the future state for which "backward" Euler computation would return the initial state. It performs the computation not using the derivative at the current time-step, but using the predicted derivative for the next time-step. Besides the inverse Euler method, other, more accurate higher-order implicit methods exist, such as the inverse Midpoint method, which remains quite simple but exhibits some instability problems. A simple solution is to interpolate between the equations of the Euler and Midpoint methods, as proposed by Volino et al [VOL 00]. Higher-order methods, such as the Rosenbrook method, however do not exhibit convincing efficiencies in the field of cloth simulation. Multi-step methods, which perform a single-step iteration using a linear combination of several previous states, are other good candidates for a good accuracy-stability compromise. Among them, the second-order Backward Differential Formula (BDF-2) has shown some interesting performances, as used by Eberhardt, Hauth et al [EBE 00] [HAU 01] and Choi et al [CHO 02].

Whatever variation chosen, the major difficulty in using implicit integration methods is that they involve the resolution of a large and sparse linear equation system for each iteration, constructed from the Jacobian matrix of the particle forces against their position and speed. A commonly used simplification involves linearization of the mechanical model so as to obtain a linear approximation of the matrix that does not evolve along time, and on which initial construction and pre-processing allows efficient resolution method to be used, as for example like Kang et al [KAN 00], or even the matrix inverse to be pre-computed as done by Desbrun et al [DES 99]. A further simplification is to suppress completely the need of computing the matrix using an adapted approximation embedded directly in an explicit iteration. A big drawback of all these methods results from the approximation of the matrix that cannot take into account the nonlinearities of the model (mostly those resulting from the change of orientation of the surface elements during the simulation). While this is acceptable for draping applications, animations behave usually poorly because of excessive numerical damping, which also increases as the time-step becomes large.

The best numerical method for actually resolving the linear system seems to be the Conjugate Gradient method, as suggested by Baraff et al [BAR 98], with various variations and preconditioning schemes depending on how the mechanical model is formulated and geometrical constraints of the cloth integrated.

Most models using implicit integration schemes restrict themselves to using spring-mass systems, as their simple formulation eases the process of defining the linear system to be resolved. However, implicit integration methods can also be used for integrating accurate surface-based particle systems as the one described above, from derivation of the particle force expressions relatively to the particle positions and speeds. This in quite simply integrated into the implicit formulations described by Volino et al [VOL 00], and extended toward other advanced methods as by Hauth et al [HAU 01]. These formulations actually blur the line between particle systems and finite element methods, as the described particle system is indeed a first-order finite element method where the implicit resolution scheme corresponds to the energy minimization scheme of finite elements and the build of the linear system matrix to the assembly process of elements into the global system to be resolved. This is a key idea to design a new system which combines the accuracy of finite elements with the efficiency of the techniques used for particle system.

4. Real-Time Garment Animation

Real-time garment animation poses the challenging problem of how to perform very fast methods for the mechanical computation and collision detection. Accuracy has to be given up in favour of quicker methods that take advantage of geometrical approximations and contextual simplifications.

4.1. Physically-based Simulation

Implicit integration [BAR 98] [CHO 02] and speedoptimized derivatives [DES 99] [MEY 00] allow fast simulation of mechanical properties of cloth. However, the computation speed still remains slow for complex garments, and these methods are still limited by the maximum number of polygons they can animate in real-time.

In the specific of area optimizations for mechanical behaviour, James et al. [JAM 99] have worked on real time simulation. Their paper describes the boundary integral equation formulation of static linear elasticity as well as the related Boundary Element Method discretization technique. Their model is not dynamic, but rather a collection of static postures, limiting its potential applications. Debunne et al. [DEB 00] have also recently introduced a technique for animating soft bodies in real time. However, their method works on volumetric meshes and is therefore not applicable to thin objects such as cloth.

4.2. Collision detection

Collision detection is another bottleneck in the speed of cloth simulation. Besides the traditional methods, specific optimizations intend to address the problem of real-time simulation. For instance, Vassilev et al. [VAS 01] propose to use z-buffer for collision detection in order to generate depth and normal maps. The computation time of their collision detection does not depend on the complexity of the body. However, the maps need to be pre-computed before simulation, also restricting the real-time application.

4.3. Geometric approaches

Some other researchers have used geometrical approaches [WEI 86] [AGU 90] [HIN 90] [NGG 95]. Geometrical models do not consider the physical properties of the cloth, therefore providing techniques that produce fast results. However, these techniques are not able to reproduce the dynamics of clothes. Moreover, geometrical techniques require a considerable degree of user intervention. They can be regarded as a form of advanced drawing tools.

4.4. Hybrid Approaches

Hybrid approaches aim to combine nicely physically based deformation and geometric deformation. The idea of hybrid approach stems from the following observation: physically based simulations are slow to compute but produce realistic results while simulations based on geometric method are much faster but not really suitable to animate full clothes. By combining advantages of the both approaches, one can expect to have acceptable results within moderate computation time. In most cases, physically based models are used to compute the global movements of garments and the details such as wrinkles are generated with geometric models.

Kang et al. [KAN 01] [KAN 02] improved the visual quality of the garments of small number of polygons by tessellating the triangles. With a cubic spline curve, their tessellation algorithm can simulate the wrinkles. Oshita et al. [OSH 01] use a similar approach to Kang et al. These both methods are mainly applicable to flat surfaces where physical simulation can be done with a relatively small number (<1,000) of polygons. However, highly curved surfaces, such as sleeves, need to be simulated with a higher number of polygons.

Recently, Hadap et al. [HAD 99] have proposed to simulate the cloth wrinkles with bump map. The global movements of the mesh are simulated with a particle system. The bump/displacement map is the product of a modulation map and a wrinkling pattern. The modulation map is generated by computing the local deformation of cloth triangles and the wrinkling pattern is designed by the CG artist.

5. Garment Design and Simulation

A Since the first developments to produce simulated garments on virtual characters [LAF 91] [CAR 92], cloth simulation and garment animation has made its way not only in computer research (fig.3) [VOL 97], but also into commercial products aimed both for 3D computer design and the garment industry.

Two kinds of products ace currently available: Those oriented for general cloth simulation and animation, and those specialized for draping and fitting garment models on virtual mannequins. The first category offer tools for simulating any kind of deformable surface mechanically. They usually offer a simple mechanical model containing only the basic mechanical parameters of cloth (stiffness, viscosity, bending, and gravity) modelled as a spring-mass particle system and simulated using state-of-the-art integration techniques. They allow the computation of realistic cloth animation, but do not provide any tool for designing garments. The also offer general collision detection schemes for interaction with any other objects. These tools are usually integrated as plug-ins into 3D design and animation frameworks. Among the main products, there is MayaCloth integrated into Maya [ALI 04], Reactor [DIS 04], Stitch [DIG 04], SimCloth [CHA 04] [MAX 04].

The second category focuses on garment draping on virtual mannequins for visualization (virtual fashion, web applications) and prototyping purposes (garment design applications). The CAD applications specialize the simulation on pattern assembly and garment draping using accurate mechanical models of fabrics, while the visualization application take advantage of geometric techniques for generating quickly realistic dressed mannequins out of design choices. Both use pattern models imported from professional pattern design tools (Gerber, Lectra, Investronica). These tools also usually provide a standalone environment for setting up the simulation and visualizing the results. Among them, there is Toyobo DressingSim [DRE 04], Browzwear V-Stitcher [BRO 04], or web applications such as FitMe.com [FIT 04] and MIRALab's Virtual Try-On [MIR 04].



Figure 3: Virtual fashion.

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Prepositioning, Physical Models, Numerical Solvers for Cloth Animations and Virtual Cloth Design

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Abstract

Prepositioning is an important tool to obtain a good initial position for the subsequent simulation of the garment drape. Here, the planar patterns are mapped onto bounding volumes providing a collision-free initial state for subsequent sewing and simulation. Material parameters, their modelling into physical models and numerical solvers for the appearing differential equations are the key modules for every physical cloth simulation. Application areas range from pure visual effects for film and entertainment industry to demanding virtual try-on scenarios, where in all cases a high degree of physical realism combined with fast computation times is needed. Therefore, research in cloth animation has focused on improving realism as well as computational speed, and significant advances have been made over the last years. In this part of the tutorial, we will discuss physical models for cloth ranging from simple mass-spring systems to continuum-based particle systems and fast finite element solutions. Then, we will describe several explicit and implicit time integration schemes and compare them with respect to stability and performance.

Finally, we will discuss interaction techniques to provide to the user a tailor and fitting room in order to manipulate garments in three-dimensional space.

1. Introduction and Overview

The area of physically based modelling is situated at the intersection of computer science, mathematics, physics, and material sciences. The animation of cloth is a particularly interesting application of physically based modelling, because it aims at fast animation solutions for rather difficult physical problems. Moreover, it addresses one of the major difficulties in creating realistic scenes with virtual actors.

The challenge of computer animation is to extract physical models for complex structures such as textiles, approximate them efficiently by mathematical algorithms, and run fast simulations with intelligent numerical methods. The range of methods proposed in literature is quite large. The techniques vary from simplified methods designed specifically for real-time applications to sophisticated methods that were designed to reproduce measured material properties. Due to improved algorithms and faster computers, it becomes possible more and more to use advanced physical models and still achieve fast animations.

In this part of the tutorial, we will first discuss several

physical models that have been employed for cloth animation in the past, ranging from discrete mass-spring and particle systems to finite element solutions for continuous cloth models. Then, we will explain explicit and implicit numerical methods for the solution of the arising ordinary differential equations.

2. Prepositioning of garment patterns

The goal of this tool is to provide an automatic close initial position of virtual garments on 3D characters. This prepositioning step is an absolutely vital component in the simulation of clothing. The cloth patterns (e.g. of a skirt, a pair of trousers or a shirt) are usually represented by planar outlines in form of borderline curves. To dress a figure with these patterns, further information on seams connecting the patterns is necessary. Out of these inputs the complete garment can be constructed. At the same time the positioning of the patterns around a 3D-character is relevant for fitting and virtual try-on. To this end we have to pay attention to prevent penetrations between the cloth patterns and their connecting

seams with the body of the character. This can be accomplished by projecting the patterns on bounding volumes. A good approximation of the character and hence a good positioning of the garment patterns is also reflected in the quality of the subsequent cloth simulation. The farther a garment lies away from the character the higher forces appear during the sewing and simulation process.

The prepositioning process thus replaces the try-on process in reality. However it is much less expensive to compute a preposition of the patterns than a real dressing process because of the complicated simulation of forces which would be required.

3. Physical Models

Models for the draping of cloth have been designed with different objectives. A common objective in computer graphics is to generate convincing and visually pleasing pictures and films. For that purpose, physics may be ignored or simplified to a certain extent. A different (engineering) objective is to preserve measured physical properties in order to map real materials onto a simulated cloth. This, for instance, is indispensable in commerce applications, in which a customer selects clothes based on a simulation and relies on the material properties for the fit. Additionally, in computer graphics, this also should lead to an animation that is fast and allows interaction with a complex scene.

In the following, we will start with relatively simple massspring and particle systems. After that we will describe how discrete and continuous models aim at preserving real material properties.

$\mathbf{s}(u,v)$	deformed surface
$\mathbf{r}(u,v)$	(local, partial) rest state of surface
$\mathbf{d}(u,v)$	displacement
\mathbf{x}_i	particle positions
\mathbf{v}_i	particle velocity
ε	strain (tensor)
σ	stress (tensor)
С	elastic tensor
D	viscous tensor
$\langle \mathbf{a}, \mathbf{b} \rangle$	scalar product of vectors a and b
Δs	Laplacian $\mathbf{s}_{xx} + \mathbf{s}_{yy} + \mathbf{s}_{zz}$
\mathbf{s}_{u}	partial derivative of \mathbf{s} with respect to u

Table 1: Notation in this section

3.1. Discrete Models

All models which we take into account have in common that they discretize the cloth by a polygonal mesh (figure 1). The vertices of this mesh are called particles or (mass) nodes. In discrete models, the mesh topology defines, how these particles interact and exert forces on one another. Given the mesh describing the cloth, forces on each particle are computed depending on its own position and velocity, and the positions and velocities of a set of particles within its topological neighbourhood. When the force function F for each particle has been determined, Newton's equation of motion yields their respective movement. The trajectory of each particle with mass m_i at position \mathbf{x}_i is computed by

$$F(x,v) = m_i \cdot \frac{d^2 \mathbf{x}_i}{dt^2} . \tag{1}$$

Here x denotes the vector containing all particle positions and v the vector of all particle velocities. Note that since particle systems already represent a discretization in space, only a system of ordinary differential equations has to be solved. The systems presented in literature differ by their methods of computing the forces.

3.1.1. Mass-spring systems

In mass-spring systems, particle interaction is solely modelled by linear springs.



Figure 2: Provot's mass-spring system with (1) structural springs, (2) shear springs, and (3) bending springs

Provot [Pro95] proposes a mass-spring system for textiles and uses a rectangular mesh in which the particles are connected by structural springs to counteract tension, diagonal springs for shearing, and interleaving springs for bending as shown in figure 2. Forces by linear springs between two particles at \mathbf{x}_i and \mathbf{x}_j are given by

$$F_{ij}^{e}(x) = k_{ij}(\|\mathbf{x}_{i} - \mathbf{x}_{j}\| - l_{ij})\frac{\mathbf{x}_{i} - \mathbf{x}_{j}}{\|\mathbf{x}_{i} - \mathbf{x}_{j}\|},$$
(2)

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Figure 1: *Textiles are discretized by polygonal meshes. Image (a) shows a piece of cloth modelled as a quadrilateral mesh. The shirt in image (b) is represented by an unstructured triangle mesh (c).*

where k_{ij} is the elastic modulus of this spring and l_{ij} its rest length. The spring constant depends on the type of the spring. For the structural forces they are very large, whereas for the bend and shear forces the springs have small values. The different constants are related to the respective forces acting in real materials. Obviously, there is a strong interdependence between the different kinds of springs leading to nonlinear, uncontrolled effects. The diagonal shear springs, for instance, also lead to additional tension and transversal contraction.

Furthermore, in our model we need viscous forces to account for energy dissipation due to internal friction. These forces damp out kinetic energy and depend on the velocity of the object. It is a common technique to model these effects for each spring by

$$F_{ij}^d(x) = d_{ij}(\mathbf{v}_i - \mathbf{v}_j) .$$
(3)

Since these terms depend linearly on the involved velocities they are particularly well suited for the subsequent numerical integration. However, there are two major disadvantages of this simple formulation as it also penalises rigid body rotations of the respective particles. Moreover, high damping of a structural spring prevents the object from bending. Hence, this simplified damping makes the deformable object move rather stiffly. These effects are alleviated by modelling a stiff, damped spring accurately by

$$F_{ij}^{d} = d_{ij} \frac{\langle \mathbf{v}_{i} - \mathbf{v}_{j}, \mathbf{x}_{i} - \mathbf{x}_{j} \rangle}{\|\mathbf{x}_{i} - \mathbf{x}_{j}\|^{2}} (\mathbf{x}_{i} - \mathbf{x}_{j}) .$$
(4)

This is the same linear damping term (3) projected onto the direction of the spring. Unfortunately, in many cases this term complicates the implicit time integration (cf. section 4.1.3). Finally, in order to run the simulation, we only have to sum up all spring forces and plug them into equation (1).

In several approaches [Pro95, DSB99, KCC*00] for mass-spring systems another popular idea is exploited. It is motivated by a biphasic behaviour of textile materials as shown in figure 3, i.e. initially the material yields to an exerted stress easily but appears to be extremely stiff in a second phase. This effect is imitated by rather small spring constants that model the first phase. In order to model the second, almost rigid phase, the system is post-processed after each time step if the springs are elongated too much. In this process, iteratively all particle positions are modified such that a certain maximum elongation is not exceeded. Such a post-processing is justified for simple mass-spring systems that do not model specific material properties anyway. Note that the result depends on the order in which the spring elongations are corrected.



Figure 3: Biphasic spring modelled by post-correction

Recently, Choi and Ko [CK02, CK03] have proposed another technique to model a biphasic behaviour of the structural springs. Only for elongation equation (2) is applied, for compression, the structural springs remain in rest length. To this end, the mesh particles have to be replaced out of the

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plane, resulting in a buckling behaviour of the mesh. This is done by a geometric condition which is integrated into the equation of motion. By this method wrinkles appear on the simulated piece of cloth when compressed like in real cloth.

Although simple mass-spring systems do not model any specific material and are not related to measured properties of real clothes, they are capable of producing pleasing animations that are sufficient in many computer graphics applications. Visually very convincing animations based on a particle system with a sophisticated approach to handle bending behavior of cloth were presented in [CK02].

3.1.2. Representations of Cloth as discrete Mechanism

In their book on cloth modelling [HB00], Donald House and David Breen state that "*Cloth is a mechanism, not a continuous material*". Consequently, some attempts have been made to model clothes by the interaction of discrete threads that are interwoven in textiles.

Some discrete systems that have been developed in computer animation for the animation of clothes and other surfaces have the advantage that they allow fast simulations. In particular, particle systems have been successfully used for rapid animations. We can consider the quadrilateral mesh that is described by the mass nodes and structural springs in Provot's mass-spring system as a network of interwoven threads, in which one thread is given by a chain of structural springs. Different threads can interact at the mass points, where shear, bend, or other internal forces apply. In order to model the interaction of threads, more complex forces than pure spring-forces are added to the system and yield a more general particle system.

Most particle systems use potential functions for tension, bend, and shear energy. These energies are chosen to correspond to standard experiments (Kawabata [Kaw80]) to measure textile properties. Hence, the measurements from one experiment are used to model one specific energy function. All energies are modelled on a rectangular grid, where each particle interacts with its four direct neighbours. The grid is aligned with two distinct directions that are apparent in textiles (in woven materials they are called weft and warp direction). The materials show different properties in these directions and each experiment has to be carried out for both directions.

The tension energy is evaluated for each particle and depends on the four neighbours of that particle in a rectangular mesh. The tension energy of a particle at position \mathbf{x}_0 is

$$E_{t} = \sum_{i=1}^{4} \begin{cases} \frac{1}{2} C_{t_{1},i}(\|\mathbf{x}_{0} - \mathbf{x}_{i}\| - l_{i} - h_{t_{1},i})^{3} & \text{if } \|\mathbf{x}_{0} - \mathbf{x}_{i}\| \ge l_{i} ,\\ \frac{1}{2} C_{t_{2},i}(\|\mathbf{x}_{0} - \mathbf{x}_{i}\| - l_{i} - h_{t_{2},i})^{5} & \text{if } \|\mathbf{x}_{0} - \mathbf{x}_{i}\| \le l_{i} , \end{cases}$$
(5)

where l_i are the rest lengths between particles and $C_{t,i}$ and $h_{t,i}$ are material parameters. They can be used to fit measured data by a piecewise linear curve. The energy is computed

from a strain ($||\mathbf{x}_0 - \mathbf{x}_i|| - l_i$), and the strain-stress relation is modelled piecewise cubic or quintic. If we introduce a linear strain-stress relationship by replacing the exponents with 2 and set $h_{t,i} := 0$, we obtain linear spring energies related to the spring forces in equation (2).

The shear energy is modelled as

$$E_s = \sum_{i=1}^{4} \frac{1}{2} C_s (\phi_i - \frac{\pi}{2} - h_{s,i})^2$$
(6)

and the bend energy as

$$E_b = \sum_{i=1}^{2} \frac{1}{2} C_b (\psi_i - \pi - h_{b,i})^2.$$
(7)

Here C_s , C_b and $h_{s,i}$, $h_{b,i}$ are the material constants. These energies implement hinges functioning like springs that linearly depend on the shear angle ϕ and the bend angle ψ , respectively. These are the angles formed by the incident edges as depicted in figure 4.



Figure 4: Shear and bend energy in a particle system (image by Eberhardt [EWS96])

All derived energies are combined to compute the final forces to be plugged into equation (1):

$$F = -\operatorname{grad}(E_t + E_s + E_b + E_{\operatorname{external}})$$

In this section, only elastic forces have been discussed. Viscous forces should be modelled in the fashion of section 3.2.7.

3.1.3. Triangular meshes

Until now, we only have considered particle systems based on rectangular meshes. Triangular mass-spring systems are widely used as well and can be constructed with almost the same set of forces. However, their physical properties are hard to control and depend on the topology of the mesh. Furthermore, they usually show a very strong transversal contraction. This motivated Volino [VMT97, VMT00b] to extend the concept of triangular mass-spring systems. In a triangular mesh the deformation of each face is uniquely determined by the elongation of its edges. Forces acting on each of its particles can be formulated depending only on these (vectorial) elongations. This results in a particle system in which the forces on one particle do not only depend on adjacent edges but on the elongations of all edges of all faces incident to the considered particle. The coefficients of these dependencies are the material constants and allow a flexible modelling of the physical properties. This model was extended to modelling measured physical data in [VMT05]. In a particle system model viscoelastic behaviour is relates to cloth triangles through simultaneous interaction between its three particles. The strain-stress relation is defined by polynomial spline approximations of the measured strain-stress curves.

The cloth simulation by Baraff and Witkin [BW98] also is based on a particle system for triangle meshes, however without the objective of fitting to real material data.

3.2. Continuous Models

Although clothes are not homogeneous, continuous objects, modelling them as discrete mechanism involves complications. As we cannot represent each single thread in a textile by an edge in the mesh, we have to choose a certain resolution of the object. If we want to be independent of this resolution, we need to represent a patch of textile as a continuous material, which allows us to use low resolution models without loosing basic material properties.

From a continuum model a consistent discretization can be derived. Consistency here means that the computed solution of the discrete problem converges to the accurate solution for the continuum when the resolution is increased. That allows us to switch from one resolution to another without changing the properties of the cloth. Therefore we will describe the foundations of the continuous theory, and present a particle system that can approximate this theory. Moreover, we describe a linear finite element solution for cloth modeling which provides about the same performance as particle systems if implicit time integration methods (section 4.1.3) are employed.

3.2.1. Descriptions of Strain

Continuum mechanics is the standard theory to describe and model deformable objects, and the following elaborations are based on several text books [Bra97, Cia92, LL89, SK95, BW00].

The basic quantities of continuum mechanics are *strain*, which is a dimensionless deformation noted by ε , and *stress*,

which is a force per length for surfaces or per area for volumes and is denoted by σ . In the case of a one-dimensional spring, these entities are scalars: the strain is its elongation per length, while the stress is the spring force. In the case of surfaces or volumes, these entities are tensors.

Surfaces are more complicated than a one-dimensional spring, and the description of strain is more involved. Textiles can be described as regular surfaces (in the sense of differential geometry [DoC76]). The deformation of a regular surface embedded in \mathbf{R}^3 is described by a strain tensor with respect to a certain undeformed reference state. In this equilibrium state, denoted by \mathbf{r} , the object is not deformed, and the elastic energy is zero. Let \mathbf{r} be parametrised over a domain $U \times V$. Under forces the rest state deforms to a state $\mathbf{s}(u, v)$. Its displacement is a mapping \mathbf{d} defined by $\mathbf{d}(u, v) = \mathbf{s}(u, v) - \mathbf{r}(u, v)$ as depicted in figure 5.

The difference of the first fundamental forms I_s and I_r of the current state and the equilibrium state of the object describes the in-plane strain and defines a nonlinear strain tensor [Kli89]

$$\tilde{G} = \frac{1}{2} (I_s - I_r) =$$
(8)
$$\frac{1}{2} \begin{pmatrix} \langle \mathbf{s}_u, \mathbf{s}_u \rangle & \langle \mathbf{s}_u, \mathbf{s}_v \rangle \\ \langle \mathbf{s}_u, \mathbf{s}_v \rangle & \langle \mathbf{s}_v, \mathbf{s}_v \rangle \end{pmatrix} - \frac{1}{2} \begin{pmatrix} \langle \mathbf{r}_u, \mathbf{r}_u \rangle & \langle \mathbf{r}_u, \mathbf{r}_v \rangle \\ \langle \mathbf{r}_u, \mathbf{r}_v \rangle & \langle \mathbf{r}_v, \mathbf{r}_v \rangle \end{pmatrix}.$$

For planar surfaces, the deformation is defined uniquely by the difference of the metrics of these states. As a piece of cloth is a surface embedded in three-dimensional space, the curvature tensors (second fundamental forms) have to be taken into account as well. Terzopoulos and Fleischer [TF88] developed a model for animated surfaces based on the energy due to these tensors.

Commonly, the rest state \mathbf{r} is assumed to be the identity mapping. Then \tilde{G} coincides with Green's strain tensor



Figure 5: The reference configuration: the rest state is parametrised by a mapping **r** on a space $U \times V$. By deformation **d** it transforms into the deformed (strained) configuration, which is parametrised by the mapping **s**.

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Green's tensor, unfortunately, is nonlinear and yields fourth order terms in the energy formulation, which are computationally very costly and lead to various numerical problems. Linear elasticity theory would allow much faster animations. It makes use of the linear approximation of Green's tensor, called Cauchy's strain tensor. It is obtained by neglecting terms of order higher than one in the displacement **d** in the components of Green's tensor, here for two dimensions:

$$\langle \mathbf{s}_{u}, \mathbf{s}_{u} \rangle - 1 = \langle \mathbf{e}_{1} + \mathbf{d}_{u}, \mathbf{e}_{1} + \mathbf{d}_{u} \rangle - 1$$

$$= 2 \langle \mathbf{d}_{u}, \mathbf{e}_{1} \rangle + O(\mathbf{d}^{2})$$

$$\langle \mathbf{s}_{v}, \mathbf{s}_{v} \rangle - 1 = \langle \mathbf{e}_{2} + \mathbf{d}_{v}, \mathbf{e}_{2} + \mathbf{d}_{v} \rangle - 1$$

$$= 2 \langle \mathbf{d}_{v}, \mathbf{e}_{2} \rangle + O(\mathbf{d}^{2})$$

$$\langle \mathbf{s}_{u}, \mathbf{s}_{v} \rangle = \langle \mathbf{e}_{1} + \mathbf{d}_{u}, \mathbf{e}_{2} + \mathbf{d}_{v} \rangle$$

$$= \langle \mathbf{d}_{u}, \mathbf{e}_{2} \rangle + \langle \mathbf{d}_{v}, \mathbf{e}_{1} \rangle + O(\mathbf{d}^{2}) , \qquad (9)$$

where $(\mathbf{e}_1, \mathbf{e}_2)$ is the Cartesian basis of \mathbf{R}^2 . Thus, Cauchy's tensor can be written as

$$\boldsymbol{\varepsilon} = \begin{pmatrix} \mathbf{d}_u^1 & \frac{1}{2}(\mathbf{d}_v^1 + \mathbf{d}_u^2) \\ \frac{1}{2}(\mathbf{d}_v^1 + \mathbf{d}_u^2) & \mathbf{d}_v^2 \end{pmatrix},$$

where the superscripts denote the vector components. Linear elasticity is based on this linearised strain tensor and yields much simpler formulations. In particular, it results in linear partial differential equations. These linear equations are widely used in engineering and lend themselves to finite element formulations very easily.

The strain tensor of linear elasticity, as we have seen, is derived from Green's strain tensor by linearisations in the deformations, i.e. the displacement **d** (figure 5) is assumed to be small, and all terms of order two or higher in **d** are neglected in the strain tensor. For this reason the linear theory is not appropriate for highly flexible objects like clothes, it only applies to small displacements. It is therefore not invariant under rigid body rotations and leads to unphysical behaviour if the object or a part of it is rotated. As animated surfaces can bend strongly, the displacements become very large, although the respective deformations remain small. Thus, a linearisation with respect to a global reference frame is not applicable to cloth animation. However, we will see later (in sections 3.2.5 and 3.2.6) that a linearisation with respect to a local reference frame is feasible.

3.2.2. The equation of motion

So far, we only have dealt with the description of strain. In linear elasticity, also the relation between the stress tensor σ and strain ε is assumed to be linear, and the dependence is given by the elastic tensor *C*. This is formulated by Hooke's law:

$$\sigma_{ij} = C_{ijkl} \, \varepsilon_{kl} \, . \tag{10}$$

C is a symmetric rank-4 tensor containing the material properties. Here, symmetry means $C_{ijkl} = C_{klij}$ as well as $C_{ijkl} =$

 C_{jikl} . Hooke's law is used to compute the stress tensor in the equation of motion of a continuous elastic material:

$$\rho \frac{\partial^2 s}{\partial t^2} - \operatorname{div} \boldsymbol{\sigma} = f, \qquad (11)$$

where ρ is the mass density and the divergence of the stress σ yields the force density due to the interior energy of the elastic object. *f* denotes an external force density (e.g. the gravity force density $\rho \cdot g$). Equation (11) is a partial differential equation (PDE) that has to be solved over time and the parameter domain of the surface. A standard procedure is to semidiscretize the system in space with finite differences or finite elements. This eliminates all spatial derivatives in the equation and reduces the PDE to an ordinary differential equation (ODE) in time *t* that can be solved by a suitable integration method. This way, we reduce the PDE (11) to the ODE of the form (1).

3.2.3. Bend forces

Bend forces cannot be derived from the standard strain tensor because they are out-of-plane forces and arise only due to a volumetric property of the (thin) object (an ideal surface does not exhibit bending forces). Hence, there are basically two possibilities of deriving bending forces. First, we can simply add some bending forces to the in-plane forces, derived for instance from a Laplacian formulation or from a thin plate energy. Second, the cloth can be modelled as a thin volumetric object. In this case, shell theory provides the appropriate mechanism. Shells are thin objects which span over surfaces. The shell is parameterised by a mid-surface **s**:

$$\mathbf{x}(u, v, w) = \mathbf{s}(u, v) + w\mathbf{d}(u, v),$$

i.e. the object position is described by the mid-surface parameters u, v and the height over the mid-surface w in the direction of a director **d** that has unit length.

From this description a three-dimensional strain and stress tensor is derived. Shells comprise (in-plane) membrane forces as well as bending forces. Some authors [EDC96, JU00, GKS02] model textiles and other thin deformable objects accurately by a nonlinear shell theory. In order to solve the resulting nonlinear PDE's, the system is discretized by finite elements, and Newton's method is used to compute an equilibrium solution.

3.2.4. Description of Clothes in a Linear Theory

A characteristic property of textiles is orthotropy. Linear, orthotropic materials possess two orthogonal symmetry axes in continuum mechanics. This reduces the number of free elastic material constants (entries in the elastic tensor) to four related to in-plane deformations. The symmetry axes or directions in a woven material are the weft and warp directions. The textiles show very different physical behaviour in these directions, and this characterises the materials. Therefore, material measurements are carried out for the two directions independently [Kaw80]. The two Young moduli $k_1 := C_{1111}$ and $k_2 := C_{2222}$, the shear modulus $\mu := C_{1212} = C_{2121} = C_{2112} = C_{1221}$, and the parameter $C_{1122} = C_{2211}$, which controls the transverse contraction, have to be taken into account as elastic constants to model the in-plane stress. Additionally, the bending moduli B_1 and B_2 describe the curvature elasticity in the weft and warp directions.

Unfortunately, the strain/stress relation in textile materials is highly nonlinear, and approximations to these nonlinear properties have been an issue in computer animation. Generally, the nonlinear stress-strain relation is modelled by a piecewise linear function. This can be achieved by updating the the elastic tensor according to the current slope of the strain/stress curve [EWS96]. The updating is carried out before each time step. Another approach [VMT05] consists in the approximation of the measured stress/strain by a polynomial spline already mentioned in section 3.1.3.

3.2.5. Continuous Theory and Particle Systems

As we have seen, Cauchy's stress tensor is not valid for large displacements. Green's strain tensor, however, impedes a rapid numerical solution and makes implicit time integration very difficult (see section 4).

In this section, we will show how the diagonal components of the strain tensor can be linearised locally to alleviate the solution of the ODE. From this model, we can derive a particle system that models continuous objects by a finite difference discretisation. This elastic model will be summarised here briefly. For details we refer to the full article [EGS03].

Given the deformed surface s(u, v), we approximate the stress tensor by

$$\boldsymbol{\sigma} = \begin{pmatrix} k_1(\|\mathbf{s}_u\| - 1) & \frac{1}{2}\mu < \mathbf{s}_u, \mathbf{s}_v > \\ \frac{1}{2}\mu < \mathbf{s}_u, \mathbf{s}_v > & k_2(\|\mathbf{s}_v\| - 1) \end{pmatrix}, \quad (12)$$

where k_1 , k_2 , and μ are the elastic material constants. This tensor is plugged into the equation of motion for continuous objects (11). In particular, in the strain tensor the dominating diagonal components have been linearised with respect to a local rest frame. Thus, the numerical solution is alleviated. From the diagonal components of the stress tensor we derive the following expression for the stress by finite difference discretization:

$$\sigma_{ii} = \frac{k_{1,2}}{l^2} (\|\mathbf{x}_i - \mathbf{x}_j\| - l),$$

where *l* is the rest distance between the particles at \mathbf{x}_i and \mathbf{x}_j . Linear spring forces result from approximating the divergence of the diagonal entries in the stress tensor

$$f_{i,j} = \frac{k_{1,2}}{l^2} \left[(\mathbf{x}_i - \mathbf{x}_j) - l \frac{\mathbf{x}_i - \mathbf{x}_j}{\|\mathbf{x}_i - \mathbf{x}_j\|} \right],$$
(13)

Note that the $f_{i,j}$ are actually force densities as we have continuous objects.

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Additionally to the stretch forces, shear forces can be approximated similarly, and bend forces can be derived from a thin plate energy (see [EGS03]).

For rectangular meshes it can be verified that this particle system approximates the accurate continuum model with linear precision, i.e. we have convergence to the accurate solution when the resolution is increased. For general quadrilateral meshes, the solution still converges, but does not possess the approximation property.

3.2.6. A Linearised Finite Element Model

The approach described in the last section works only for regular quadrilateral meshes. Therefore, a linearised finite element model for cloth which works for arbitrary unstructured triangle meshes has been developed [EKS03]. It is particularly designed for numerically stiff materials such as textiles because it yields linear equations in each time step and allows fast time stepping in an implicit integration method (cf. section 4.1.3).

For the moment we assume that all forces and displacements are in the (x,y)-plane, and that the small displacement assumption holds. For this case, we state the finite element formulation for linear elasticity as presented in several text books (e.g. [BW00]). Given the surface $s(u_1, u_2)$, we denote the linear form functions of the linear finite element space by $N_j(u_1, u_2)$. For each triangle $T_m(a, b, c)$ we compute the form factors $\frac{\partial N_j}{\partial u_i}$, j = a, b, c, i = 1, 2 in the rest state (these form factors replace the rest lengths of the springs in a massspring system). Also we compute the triangle areas Ω_{T_m} in the rest state. Then we construct the stiffness matrix as follows. For each triangle T_m and each vertex pair (a, b) of this triangle, we define a submatrix $K_{ab}^m \in \mathbb{R}^{2\times 2}$

$$\begin{bmatrix} K_{ab}^m \end{bmatrix}_{ij} = \left(\sum_{k,l=1}^2 \frac{\partial N_a}{\partial u_i} C_{ikjl} \frac{\partial N_b}{\partial u_j} \right) \Omega_{T_m}, \tag{14}$$

where C is the elastic tensor that contains all elastic material properties.

The stiffness matrix $A \in \mathbb{R}^{3n \times 3n}$, where *n* is the number of vertices, is assembled from all matrices K_{ab}^m for all triangles. It consists of $n \times n$ submatrices that are computed as follows:

$$A]_{ab} = \begin{pmatrix} \begin{bmatrix} \sum K_{ab}^m \\ m|T_m \in \Gamma(a,b) \\ 0 & 0 \end{bmatrix} & \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \in \mathbb{R}^{3 \times 3}$$

Here $\Gamma(a, b)$ is the set of all triangles containing vertices a, b.

We denote the displacement, position, and velocity vectors that store the values of all *n* vertices by $d, x, v \in \mathbb{R}^{3n}$, respectively. The force vector is then computed by

$$F(x) = Ad = A(x - \bar{x}),$$

where *x* is the vector of the vertex positions in the deformed

state and \bar{x} the vector of positions in the rest state. This force formulation is valid as long as the surfaces remain in the (x,y)-plane.

As mentioned previously, large rotations prevent the usage of the linear strain formulation. Hence, in each time step we are going to remove these rotations before computing the deformation forces (similar ideas have been used for threedimensional deformable objects in [MDM*02] and [HS04]). This is equivalent to the construction of a rotated rest state and the linearisation with respect to this rest state. The key here is the polar decomposition of the deformation gradient. The deformation gradient J can be decomposed into a rotation R and a pure deformation U with

$$J = R \cdot U.$$

In the two-dimensional case the polar decomposition can be calculated directly via an eigenanalysis with respect to J and is carried out as follows:

- 1. Compute $U^2 = J^t J$
- 2. Compute the eigenvalues λ_1,λ_2 and eigenvectors $\boldsymbol{v}_1,\boldsymbol{v}_2$ of U^2
- 3. Compute $U = \sqrt{\lambda_1} \mathbf{v}_1 \mathbf{v}_1^t + \sqrt{\lambda_2} \mathbf{v}_2 \mathbf{v}_2^t$ 4. Compute $R = JU^{-1}$

Since we use linear finite elements, the matrix J is constant on each triangle. Hence, the polar decomposition gives us a unique rotation R^m for each triangle T_m . This rotation is used to move the deformed triangle back to a planar state in the (x,y)-plane, compute the elastic forces with linear elasticity there, and finally rotate the computed forces back in 3D space. For that purpose, we have to transform the displacement vectors before the multiplication with the local stiffness matrix. The force vector that is the result of the multiplication has to undergo the inverse transformation. One local stiffness matrix for edge (a,b) computes forces as follows:

$$F(x) = R^m K^m_{ab} (R^m)^t \mathbf{x}_a - R^m K^m_{ab} \bar{\mathbf{x}}_a$$

Note that the planar rest state position $\bar{\mathbf{x}}_a$ is already in 2D space and only the results of the multiplication need to be rotated to 3D space. Hence, we define the transformed local stiffness matrices as

$$Q^m = R^m \cdot K^m \cdot R^t_m$$

$$\bar{Q}^m = R^m \cdot K^m ,$$

where $R^m \in \mathbb{R}^{3 \times 2}, Q^m \in \mathbb{R}^{3 \times 3}, \bar{Q}^m \in \mathbb{R}^{3 \times 2}$. The global stiffness matrices are then assembled from the matrices Q^m and \bar{Q}^m :

$$[A]_{ab} = \sum_{m|T_m \in \Gamma(a,b)} Q_{ab}^m , \qquad (15)$$
$$[\bar{A}]_{ab} = \sum_{m|T_m \in \Gamma(a,b)} \bar{Q}_{ab}^m .$$

The global force vector is computed by

$$F(x) = Ax - A\bar{x}$$

Similarly, bend forces can be derived from a finite element formulation of the Laplacian operator projected onto the surface normals. Then, they can be evaluated as

$$F(x) = \tilde{A}x$$

Moreover, linear viscous forces can be derived by replacing the elastic tensor C by the viscosity tensor D (cf. equation (17)). These viscous forces result in a similar matrix formulation and are evaluated by

$$F(v) = Bv.$$

For a detailed discussion of bend forces and viscosity we refer to [EKS03]. Summarizing all forces, Newton's equation of motion becomes

$$M\frac{d^2x}{dt^2} = Ax - \bar{A}\bar{x} + \tilde{A}x + Bv + f_{\text{ext}}$$

where M is the mass matrix, and f_{ext} are external forces. One time step in the animation is then computed as follows:

- 1. Compute the rotations R^m for each triangle T_m
- 2. Construct the matrices A, \tilde{A}, B and compute $\bar{A}\bar{x}$
- 3. Carry out a time step.

The achievement of this procedure is that the forces are linear, such that a system of linear equations has to be solved in each time step, if an implicit time integration method is used. For the implicit Euler method (see section 4.1.3), for instance, we have to solve

$$v_1 = v_0 + hM^{-1}(Ax_1 + \tilde{A}x_1 + Bv_1 - \bar{A}\bar{x} + f_{\text{ext}})$$

$$x_1 = x_0 + hv_1$$

with time step h. Note that the subscripts denote the time indices here. From this, we obtain the linear system

$$(M - h^2 A - h^2 \tilde{A} - hB)v_1 =$$

$$Mv_0 + h(Ax_0 + \tilde{A}x_0 - \bar{A}\bar{x} + f_{\text{ext}}).$$

3.2.7. Viscosity and Hysteresis in Textiles

Regarding the nonlinear behaviour of cloth figure 6 shows a typical strain/stress curve that can be measured when a textile patch is stretched and relaxed. Hysteresis effects appearing as in this figure are due to energy dissipation. When the material deforms under external forces, the strain/stress relation is described by the upper branch of the hysteresis. When it is released and contracts again, this relationship is described by the lower branch and the area between both branches is proportional to the dissipated energy.

There are several causes for energy dissipation when modelling deformable objects. Here, we focus exclusively on intrinsic effects, i.e. those that do not depend directly on external forces like friction due to fluid flow (wind). Internal



Figure 6: Hysteresis

friction depends on the relative motion of parts of the deformable object, that is, on the strain rate. The strain rate tensor is defined as

$$\mathbf{v}_{ij} = \frac{d\mathbf{\varepsilon}_{ij}}{dt} \,. \tag{16}$$

The forces generated by internal friction are often called viscous in analogy to Newtonian fluids. A viscous stress can be computed analogously to equation (10):

$$\sigma_{ij}^{\nu} = D_{ijkl} \,\nu_{kl}, \tag{17}$$

where the "viscosity tensor" D has the same structure as the elastic tensor C. The viscous stress is added to the purely elastic stress. In most implementations the viscosity tensor D is chosen constant and proportional to the elastic tensor C. This produces a material called a Kelvin-Voigt solid, the simplest viscoelastic solid. Note that the spring damping forces as stated in equation (4) can be derived from this model. In order to fit the typical behavior of cloth, a more complex and visco-elastic model has to be chosen. For instance a constant Q model allows to reproduce a measured hysteresis curve [GEHB01].

4. Numerical Simulation

In this section, we will describe several explicit and implicit time integration methods, in parts following Hauth [HES03]. A further in-depth comparison of integration methods for cloth simulations has been presented by Volino and Magnenat-Thalmann [VMT01, VMT05].

4.1. Methods for Numerical Integration

As we have seen in the previous section mechanical systems are often given as a second order ordinary differential equation accompanied by initial values

$$x''(t) = f_v(t, x(t), x'(t)) ,$$

$$x(t_0) = x_0, x'(t_0) = v_0 .$$
(18)

The differential equation can be transformed into a first order system by introducing velocities as a separate variable:

$$\begin{bmatrix} x(t) \\ v(t) \end{bmatrix}' = \begin{bmatrix} v(t) \\ f_v(t, x(t), v(t)) \end{bmatrix}, \begin{bmatrix} x(t_0) \\ v(t_0) \end{bmatrix} = \begin{bmatrix} x_0 \\ v_0 \end{bmatrix}.$$
(19)

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For the next few sections it will be convenient to write this ODE in the more abstract form

$$y'(t) = f(t, y(t)), \quad y(t_0) = y_0,$$
 (20)

before we will come back to the special setting (19) for eventually gaining computational advantages.



Figure 7: Solutions of example 1 for $\lambda = 2$ (dashed) and $\lambda = -15$ (solid)

Throughout the following discussion we will use the following examples:

- 1. $y' = \lambda y, y(0) = 1$ with $\lambda = 2, -15$ for $t \in [0, 1]$ (figure 7).
- 2. The overdamped wave equation $y'' = \lambda/2y + \lambda y'$ with $\lambda = 5$ for $t \in [0, 10]$ and starting values y(0) = 0, y'(0) = 1. It has the analytical solution $y(t) = 1/15\sqrt{15}e^{1/2}(-5+\sqrt{15})t 1/15\sqrt{15}e^{-1/2}(5+\sqrt{15})t$
- 3. This example is based on a simple mechanical system (figure 8(a)): A particle *p* with mass *m* connected to the origin using a spring with stiffness *k*, damping *d* and rest length *l*₀, is pulled down by gravity. This setting is described by the ODE

$$\frac{d^2z}{dt^2} = \frac{k(l_0 - z)}{m} - \frac{d}{m}\frac{dz}{dt} + g_z .$$
(21)

We set the parameters m = 0.1, k = 100, $l_0 = -1$, d = 1, $g_z = -10$, $z_0 = -1$, $v_{0,z} = -5$ and simulate the interval $t \in [0, 2]$ (figure 8(b)).

4.1.1. Explicit methods

The oldest and most simple method of integration is the so called forward or explicit Euler method. Time is discretised into steps of length h. To get a formula for advancing a timestep, h the differential quotient on the left hand side of (20) is replaced by the forward difference quotient

$$\frac{y(t+h) - y(t)}{h} \approx y'(t) = f(t, y(t)) .$$
 (22)

Thus we get the integration formula for advancing a single timestep

$$y(t+h) = y(t) + hf(t, y(t))$$
. (23)



(a) The mechanical system of example 3.



(b) Exact solution of example 3: z (solid), z' (dashed), and an implicit Euler solution (see below) for large timesteps.



Iterating this method gives a sequence of numerical approximations Y_n . Geometrically this method can be interpreted as straightly following the tangent of the solution at the actual state and then recalculating the slope for the next step.

There are several criteria for evaluating an integration method:

- convergence
- accuracy
- stability
- efficiency

Convergence means that for $h \rightarrow 0$ the numerical solutions Y_n meet the analytical. All useful methods must be convergent, so we won't discuss non-convergent methods or criteria for convergence. More interesting is the accuracy or order of a method. By this we mean how fast a method converges for $h \rightarrow 0$, or with other words how accurate the solution is for

a given *h*. By using a taylor expansion for the exact solution after a single timestep

$$y(t+h) = y(t) + hy'(t) + h^2/2y''(t) + O(h^3)$$
(24)

we find that for the numerical approximation Y_1 produced by an explicit Euler step

$$y(t_1) - Y_1 = O(h^2)$$
. (25)

If we continue the method using the *numerical solution* Y_1 as a starting value for the next timestep we lose [HW96a] a power of *h* for the global error

$$y(t_n) - Y_n = O(h) . (26)$$

This means that the explicit Euler method converges linearly or has *order* 1. We will analyse stability and efficiency of the method later.

As a next step we will introduce methods of higher order. For this a centered difference estimation for y'(t+h/2) in the differencial equation (20) is used leading to

$$\frac{y(t+h) - y(t)}{h} \approx y'(t+h/2) = f(t, y(t+h/2))$$
(27)

and thus as an iteration scheme

$$Y_{n+1} = Y_n + f(t, y(t_n + h/2)) .$$
(28)

But how do we find $y(t_n + h/2)$? For an estimation we use an explicit Euler step to get

$$k_1 = Y_n + \frac{h}{2}f(t, Y_n) , \qquad (29)$$

$$Y_{n+1} = Y_n + hk_1 , \qquad (30)$$

the so called *explicit midpoint* rule. The estimation by forward Euler, although not very accurate is good enough, as the function evaluation is multiplied by the timestep to advance to the next approximation. So by a taylor expansion we find a local error of $O(h^3)$ leading to a global error of

$$Y_n - y(t_n) = O(h^2) \tag{31}$$

for the explicit midpoint rule.

Generalizing the idea of using function evaluations at *s* intermediate points $t + c_j h$ leads to the Runge-Kutta methods defined by a Runge-Kutta matrix (a_{ij}) , weights b_i , abscissae c_j and the equations

$$k_{i} = Y_{n} + h \sum_{j=1}^{s} a_{ij}k'_{j}$$

with $k'_{i} = f(t_{n} + c_{i}h, k_{i})$ for $i = 1, \dots, s$
 $Y_{n+1} = Y_{n} + h \sum_{i=1}^{s} b_{i}k'_{i}$. (32)

The coefficient set can comfortably be specified like in table 2. If the matrix (a_{ij}) is strictly upper, all inner stages k_i only depend on k_j with j < i and thus can be computed one after the other.

The most famous one is given in table (3) together with



Figure 9: Work precision diagrams for the explicit Euler, explicit midpoint and RK4 methods.

c_1	a_{11}	a_{12}	•••	a_{1s}
c_2	a_{12}	a_{22}	•••	a_{2s}
÷	:		·.	÷
c_s	a_{s1}	a_{s2}	•••	a_{ss}
	b_1	b_2		b_s

Table 2: General Runge-Kutta method



Table 3: Explicit midpoint and "the" Runge-Kutta method

the explicit midpoint rule interpreted as a Runge-Kutta method. This special method by Runge and Kutta possesses order 4.

By using algebraic relations for the coefficients, it is possible to construct explicit Runge-Kutta methods of arbitrary high order resulting in many inner stages with many func-

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tion evaluations. But for most practical applications order 4 is sufficient.

Having constructed all these methods, we now will try them on our examples. The diagrams in figure 9 were produced by solving the examples using different timesteps and measuring the number of floating point operations needed for achieving the specified accuracy when compared with the (analytical) reference solution. In the work-precision diagram the y-axis shows the error $||Y_{t_{end}} - y(t_end)||$ as a function of the required number of floating point operations. The first example with $\lambda = 2$ (figure 9(a)) shows exactly the expected behaviour: when reducing the time step and thus investing more work, the numerical solutions converge against the reference solution. Moreover the slope of the curves in the double logarithmic plot exactly match the order of the method. But in all the other examples (figure 9(b)-9(d)) this behaviour only shows up after an initial phase, where the solver produces completely wrong results. This is the point where stability comes in. We will now analyse this by using the simplest example where it occurs, i.e. example 1 with $\lambda < 0.$

4.1.2. Stability

The equation for example 1 is called *Dahlquist's test equation*

$$y' = \lambda y, \qquad \lambda \in \mathbb{C}$$
 (33)

Its exact solution to an initial value $y(0) = y_0$ is given by

$$y(t) = e^{\lambda t} y_0 . \tag{34}$$

This equation serves a tool for understanding and evaluating the stability of integration methods. We have seen, that in the damped case characterised by $\text{Re}\lambda < 0$ convergence is only achieved by very small timesteps. In this case, since the exponent is negative, the analytical solution is bounded for $t \to \infty$. Therefore a meaningful numerical method is required to deliver a bounded solution. An integration scheme that yields a bounded solution is called *stable*.

If we apply the explicit Euler's method with a fixed step size h to (33) the n-th point of each solution is given by:

$$Y_n = (1 + h\lambda)^n y_0 \tag{35}$$

For the explicit Euler scheme the numerical solution given by (35) is bounded if and only if $|1 + h\lambda| < 1$, i.e. for $h\lambda$ in the unit ball around -1. A similar analysis can be carried out for the other methods resulting in respective restrictions of the admissible step size.

This analysis explains the sharp bend in figures 9(b)-9(d). Only when the step size drops below a certain limit dictated by λ ($h < \lambda^{-1}$ in case of the forward Euler method) the numerical solutions can converge. If the damping is increased, i.e. Re $\lambda \rightarrow -\infty$, then for the explicit Euler necessarily $h \rightarrow 0$ for the solution to be stable. This means the step size is artificially limited and it cannot be increased beyond the stability limit in order to save work sacrificing some – but not all – accuracy.

4.1.3. The implicit Euler method

To construct a method that better suits our needs we go back to (20) and substitute the differential quotient by a *backward* difference quotient for y(t + h)

$$\frac{y(t+h) - y(t)}{h} \approx y'(t+h) = f(t+h, y(t+h)) .$$
(36)

This time this results in the integration formula

$$Y_{n+1} = Y_n + hf(t+h, Y_{n+1}), \qquad (37)$$

the so called backward or implicit Euler method. As its explicit variant this method can be shown to have order 1. Now the numerical solution only is given implicitly as the solution of the possibly nonlinear equation

$$Y_{n+1} - hf(t+h, Y_{n+1}) - Y_n = 0.$$
(38)

If we apply this method to the Dahlquist equation we get the recurrence formula

$$Y_n = (1 - h\lambda)^{-n} y_0$$
. (39)

Thus, the numerical solution Y_n of equation (33) remains bounded for $|(1-h\lambda)^{-1}| < 1$. If we assume $\lambda < 0$, this holds for arbitrary h > 0. Thus there is no restriction on stepsize, the method is *unconditionally stable*. Figure 11 shows the work-precision diagrams for the implicit Euler method and our examples. We observe that we never loose stability and we especially do not miss the solution by several orders of magnitude compared to the explicit methods, although we loose some accuracy when the timesteps become large.

As a practical tool for graphically visualizing the stability properties of a method we define the *stability region* S to be the set of parameters, for which the integration method yields a bounded solution:

$$S := \{z := h\lambda \in \mathbb{C} : \text{ the numerical integration} \\ \text{of equation (33) with step size } h \text{ and} \\ \text{parameter } \lambda \text{ is stable} \}.$$
(40)

Methods that contain the complete left half-plane in S are called *A*-stable or unconditionally stable. They are well suited for the stable integration of stiff equations. Obviously, the implicit Euler scheme is A–stable, whereas its explicit counterpart is not. The stability regions of all presented methods are shown in figure 10.

After reviewing the process that led us to the definition of the stability region, we can outline a more general idea that will allow us to determine easily the stability of more complex methods. The idea for analysing both Euler methods applied to (33) was to find a closed expression describing the *stability function* \mathcal{R} . This function maps the initial value y_0 to the value Y_1 , performing a single step of the method

$$\mathcal{R}: y_0 \mapsto Y_1 \ . \tag{41}$$

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Figure 10: Stability regions (shaded) of the methods.

Thus $Y_n = R(h\lambda)^n y_0$. For the explicit Euler method we found in (35)

$$\mathcal{R}(z) = 1 + z \,, \tag{42}$$

for the implicit version in (39)

$$\mathcal{R}(z) = \frac{1}{1-z} \,. \tag{43}$$

The definition for the stability region now reads

$$S = \{z \in \mathbb{C} : |\mathcal{R}(z)| < 1\} . \tag{44}$$

4.1.4. Methods of higher order

To find a higher order method, we go back to equation (28) and do not replace the y(t + h/2) but take the formula as an implicit definition of y(t + h). We get the *implicit midpoint rule*

$$Y_1 = Y_0 + hf\left(\frac{Y_1 + Y_0}{2}\right).$$
 (45)

Further on we use a simplified notation for advancing one step, writing Y_0 and Y_1 instead of Y_n and Y_{n+1} .

Alternatively the midpoint rule can be derived as a *collocation method* with s = 1 internal nodes, i.e. by constructing a polynomial interpolating the particle trajectories at a given, fixed set of *s* nodes [HW96b]. This idea allows for the construction of implicit Runge-Kutta methods with arbitrary order. In contrast to explicit methods the matrix (a_{ij}) ceases to be strictly lower triangular. These methods are computationally more expensive, so we just stick to the midpoint rule. Its stability function is given by

$$\mathcal{R} = \frac{1 + z/2}{1 - z/2}.$$
(46)

As $\mathcal{R} \leq 1$ for any $\operatorname{Re} z < 0$ the implicit midpoint rule is A-stable.

As another possible choice we now introduce *multistep methods*. They are computationally inexpensive because they have no inner stages and some of them are A-stable. A multistep method with k steps is of the general form

$$\sum_{j=0}^{k} \alpha_j Y_{k-j+1} = h \sum_{j=0}^{k} \beta_j f_{k-j+1}, \qquad (47)$$

with $f_{n+j} := f(t_{n+j}, Y_{n+j})$. Here we also have 'history points' with negative indices. The coefficient α_k is required to be nonzero; for variable time step sizes the coefficients depend on the last stepsizes, which we have omitted here for the ease of demonstration. Important special cases are the class of *Adams methods* where $\alpha_0 = \cdots = \alpha_{k-2} = 0$:

$$Y_1 = Y_0 + h \sum_{j=0}^k \beta_j f_{k-j+1}$$
(48)

and the class of *BDF–methods* (backward differentiation formulas) with $\beta_0 = \cdots = \beta_{k-1} = 0$:

$$\sum_{j=0}^{k} \alpha_j Y_{k-j+1} = h\beta_k f_1.$$
(49)

If the formula involves the right-hand side f_1 at the new approximation point Y_1 the method is said to be implicit. BDFmethods are always implicit. The coefficients can again be constructed by a collocation approach. The stability regions of implicit and explicit Adams methods are bounded and located around the origin, thus they are not interesting for large time steps.

BDF-methods were the first to be developed to deal with stiff equations and possess an unbounded stability region covering a sector within the negative complex half-plane. Therefore they are widely used today. For k + 1 points, these methods possess order k + 1 and only one nonlinear system has to be solved, whereas *s* coupled systems have to be solved for an *s*-stage implicit Runge-Kutta method.

The BDF-method for k = 1 is just the implicit Euler

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Figure 11: Work precision diagrams for the implicit Euler, implicit midpoint, BDF(2) and Verlet (whenever possible) methods. *The results for Euler are for comparison and the same as in figure 9.*

method, for k=2 the method is given as

$$Y_1 = \frac{4}{3}Y_0 - \frac{1}{3}Y_{-1} + \frac{2}{3}hf(t+h, Y_1)$$
(50)

The stability region of BDF(2) and the other implicit methods are shown in figure 10.

4.1.5. The Verlet method

As a last method we will discuss a scheme commonly referred to as leapfrog or Stoermer-Verlet method. It is especially efficient if (18) is given as the second order system

$$x''(t) = f_v(t, x(t)),$$
(51)

i.e. $f_{\nu}(t,x(t),x'(t)) = f_{\nu}(t,x(t))$. It is not applicable to general first order systems of the form (20).



Figure 12: Staggered grids for the Verlet method.

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To derive it, we use centered differences at a staggered grid (figure 12) i.e. we now approximate v at t + (2i+1)h/2 and x at t + ih by centered differences

$$\frac{v_{n+1/2} - v_{n-1/2}}{h} = f(x_n) \tag{52}$$

$$\frac{x_{n+1} - x_n}{h} = v_{n+1/2} \tag{53}$$

thus

$$v_{n+1/2} = v_{n-1/2} + hf(x_n) \tag{54}$$

$$x_{n+1} = x_n + hv_{n+1/2}.$$
 (55)

The method possesses order 2 as one can see by substituting (54) into (55) resulting in the second order centered difference

$$\frac{x_{n+1} - 2x_n + x_{n-1}}{h} = f(x_n).$$
(56)

From this equation an alternative formulation of the Verlet scheme as a multistep method can be derived

$$v_n - v_{n-1} = hf(x_n)$$
 (57)

$$x_{n+1} - x_n = hv_n, \tag{58}$$

which omits the half steps and staggered grids from above. Now for second order equations which do not possess the form of (51) one may replace $f(x_n)$ by $f(x_n, v_{n-1})$ at the expense of some stability. Correctly the replacement had to be with $f(x_n, v_n)$ but this would result in an implicit method. Now we can apply the method to examples 2 and 3.

Figure 11 shows the work precision diagrams for the implicit methods. Even for large time steps these methods give approximations to the exact solution. After a somewhat uneven convergence phase all the explicit methods converge smoothly for decreasing time step to the exact solution with a slope given by their order.

Up to now we have omitted a discussion of the stability properties of the leapfrog method. From the examples it can be observed that the method cannot be unconditionally stable. Indeed some more delicate computations show that the method only delivers a bounded solution for arbitrary h > 0for purely oscillatoric equations, i.e. for second order ordinary differential equations of the form (51) (with no damping term) and a contractive right hand side. Nevertheless the method remains well behaved in the presence of low damping. In figure 13 we applied the Verlet method to the undamped wave equation y'' = 5y and it is clearly one of the best choices over a wide range of accuracy requirements.

4.1.6. Selecting an efficient method

Which method is best for a certain application? This question is nearly impossible to answer a priori. The only choice is to try a set of methods and to evaluate which one performs best. Choosing the methods to try, though can be done based on theoretical considerations and observations of the problem at hand. A possible strategy is shown in figure 14.



Figure 13: Work precision diagram for the explicit/implicit Euler, implicit midpoint, BDF(2) and Verlet methods for the wave equation y'' = -5y without damping.

The same statement holds for predicting the efficiency of a method. Generally, implicit methods require more work per step. On the other hand one may be able to use time steps that are several magnitudes larger than the ones explicit methods would allow. Although accuracy will suffer, the integration won't be unstable. If evaluations of the right hand side function are cheap, a step with RK(4) is faster than an implicit step with BDF(4). On the other hand if it is cheap to compute a good sparse approximation to the jacobian, it may be more efficient to solve the linear system with a few cg iterations than to perform 4 full function evaluations.

4.2. Solving nonlinear systems

All implicit methods, unless they use a linear force formulation as the one described in section 3.2.6, require the solution of a nonlinear system. The implicit Euler method for example reduces our integration problem to the solution of the nonlinear system

$$Y_1 - hf(Y_1) - Y_0 = 0. (59)$$

The other methods yield a system of similar form, namely

$$Y_1 - hf(\frac{1}{2}(Y_1 + Y_0)) - Y_0 = 0$$
(60)

$$Y_1 - \frac{2}{3}f(Y_1) + \left(-\frac{4}{3}Y_0 + \frac{1}{3}Y_{-1}\right) = 0$$
(61)

for the midpoint and BDF(2) rule, respectively. This is a nonlinear system of dimension 6*N*. This system must be solved with Newton's method to allow for arbitrary step sizes independent of λ .

4.2.1. Newton's method

For the nonlinear system G(Y) = 0 we compute a numerical solution by the following algorithm:

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Figure 14: Selecting a method.

Algorithm 1: Newton's Method

(1) for k = 1, 2, ... until convergence do Compute $G(Y^{(k)})$. (2) Compute $J^{(k)} = rac{\partial}{\partial Y} G(Y^{(k)}).$ (3) Solve $J^{(k)}s^{(k)} = -G(Y^{(k)}).$ (4) $Y^{(k+1)} := Y^{(k)} + s^{(k)}$ (5) end

Applying Newton's method reduces the problem to the successive solution of linear systems. In a classical Newton method this is achieved by Gaussian elimination. This introduces a lot of non-zero elements into the factors. Although reordering techniques alleviate the effects, this approach is currently too expensive for fast cloth animations.

A lot of authors [BW98, VMT00a] use iterative methods to solve the linear system. We will also use the conjugate gradient (cg) method here to solve the linear systems in each Newton step. However, this changes the convergence behaviour of the outer Newton method, which is referred to as an inexact Newton method[Rhe98], given by algorithm 2.

Algorithm 2: Inexact Newton Method

(1) for $k = 1, 2, \dots$ until convergence do

(2) Compute
$$G(Y^{(k)})$$
.

(3)

- $\begin{array}{l} \text{Construct} \quad J^{(k)} = \frac{\partial}{\partial y}(Y^{(k)}). \\ \text{Find } s^{(k)} \text{ with } J^{(k)}s^{(k)} = -G(Y^{(k)})) + r^{(k)}, \end{array}$ (4) such that $||r^{(k)}|| \le \eta_k ||G(Y^{(k)})||$. $Y^{(k+1)} := Y^{(k)} + s^{(k)}$
- (5) end

4.2.2. Residual control

The error of the iterative solution of the linear system is formulated in terms of the residual, which is easily computationally accessible, whereas the actual error cannot be computed. The tolerance of the linear iteration is decreased proportionally to the monotonically decreasing residual of the nonlinear iteration.

An analysis of this method[Rhe98] shows that it converges under rather weak additional assumptions. If the classical Newton method converges and the scalar tolerances $\eta^{(k)}$ are uniformly bounded by an $\eta < 1$, the inexact method converges. In literature the $\eta^{(k)}$ are referred to as *forcing terms*. Note that this additional assumption is also necessary: For $\eta = 1$, $s^{(k)} = 0$ would be admissible and the iteration would stagnate.

The inexact method then at least converges linearly, whereas Newton converges superlinearly. By choosing the η_k to converge to zero sufficiently fast[Rhe98], the convergence of the inexact Newton method can be forced to have an order > 1. In a neighbourhood of the solution the convergence usually speeds up. By extrapolating the solution of the previous time step we obtain a good initial value for the new solution and the method converges quickly using the constant bound $\eta = 0.02$ without imposing a too strict tolerance on the linear solver.

4.2.3. Inexact simplified Newton methods

The efficiency of the Newton method can be further improved by another approximation. In the simplified version of Newton's method the Jacobian $J^{(k)}$ is approximated by $J^{(0)}$. Such a scheme can be rewritten in the form of an inexact Newton method, if the linear system is written as follows and J is chosen as approximation to $J^{(k)}$

$$Js^{(k)} = -G(Y^{(k)}) + (J - J^{(k)})s^{(k)} + r^{(k)}$$

=: -G(Y^{(k)}) + $\tilde{r}^{(k)}$ (62)

The residual $r^{(k)}$ is replaced by the larger $\tilde{r}^{(k)}$, which can be bounded if $J \approx J^{(k)}$. By choosing $\tilde{\eta}^{(k)}$ appropriately, the method still converges. In fact, we trade some accuracy approximating $J^{(k)}$ against accuracy in solving the linear system and up to a certain limit the method still behaves as before.

This degree of freedom can be further exploited by even not computing $J^{(0)}$ but a sparser approximation of it. In [HE01], cloth is modelled using the continuity based approach from section 3.2.5, and the choice of the approximated Jacobian is motivated by observing that the dominating stiffness is induced into the system by linear spring forces from (13). These are split into a linear and a nonlinear part.

Then, the right-hand side of the system is approximated by the linear expression

$$F_{\text{lin}}(x,v) = \sum_{j|(i,j)\in E} \left[\frac{k_{ij}}{l_{ij}^2} (x_i - x_j) + \frac{d_{ij}}{l_{ij}^2} (v_i - v_j) \right]$$
(63)

Writing this equation in matrix notation

$$F_{\rm lin}(x,v) = Kx + Dv, \tag{64}$$

the full Jacobian of the system is approximated by the Jacobian of $F_{\text{lin}}(x, v)$. Then, (62) is applied with

$$J = I - h\gamma \left(\begin{array}{cc} 0 & 0\\ K & D \end{array}\right),\tag{65}$$

where *h* is the time step and γ depends on the integration method used. This system of dimension 6*N* can be reduced to a system of dimension 3*N* by exploiting the linear relation between position and velocity [HES03].

This choice of the Jacobian has two major advantages over the full Jacobian. First, J is inexpensive to compute and only changes when either the material constants or the step size changes. Second, we reduce the entries in the Jacobian to approximately a third of the entries in the sparsity pattern of the full Jacobian. Hence an iteration of the linear solver only requires a third of the original time. Obviously this is a major speed-up for the solver. The resulting algorithm is surprisingly simple.

Algorithm 3: Inexact Simplified Newton's Method

(1) Compute $J \approx \frac{\partial}{\partial Y} G(Y^{(0)})$.

(2) for k = 1, 2, ... until convergence do

(3) Compute $G(Y^{(k)})$.

(4) Find $s^{(k)}$ with $Js^{(k)} = -G(Y^{(k)})) + r^{(k)}$, such that $||r^{(k)}|| \le \tilde{\eta}_k ||G(Y^{(k)})||$. (5) Update $Y^{(k+1)} := Y^{(k)} + s^{(k)}$

end

4.2.4. Adaptive time stepping

Newton's method can also be used to control the step size of the ODE solver. If the convergence of Newton's method is poor, the time step h is reduced such that the solution of the previous time step is a better start value for the current time step and achieves a faster convergence.

4.3. Comparison of methods

We are now able to describe many of the current approaches for cloth simulation in a unifying way.

Explicit integration methods, mostly together with a geometric post-correction step as described in section 3.1.1, have been used for instance by Provot [Pro95], Volino [VCMT95], and Fuhrmann [FGL03].

Baraff and Witkin [BW98] formulate nonlinear constraints and use their linear approximation for the construction of an implicit solution method commonly referred to as linear implicit Euler. This way the system to be solved also becomes linear and can be solved efficiently by a conjugate gradient method (see also [AB03]). This method corresponds to the solution of a nonlinear system with only one Newton iteration. Because the nonlinear part is not integrated, with high stiffness one may encounter problems which were addressed in [EEH00, HE01].

Following [BW98], implicit integration has been employed with varying physical models for instance in [VMT00a, CK02, CK03, EKS03]. Desbrun et al. [DSB99, MDDB01] also use a linear implicit method combined with Provots model [Pro95] (section 3.1.1). But instead of linearizing the whole system, they split it in a linear and nonlinear part and use a precomputed inverse of A for solving the linear part of the equations. They don't aim at solving the equation completely, as they don't integrate the nonlinear term explicitly. Instead the angular momentum is corrected to account for the nonlinear part. With this algorithm one can neither change the stepsize h nor deal with an Jacobian depending on t. Debunne et al. [DDCB01] use a simplified version of a linear implicit Euler method. To solve the differential equation arising from their geometric nonlinear Green model, they apply an implicit Euler method using a single Newton iteration. The Jacobian is approximated by the 3x3 block-diagonal of the linear Cauchy tensor, allowing a very fast inversion and application.

5. Conclusions for Physical Models and Numerical Solvers

We described physical models for cloth ranging from mass-spring and particle systems to finite element models. Moreover, we discussed and compared several explicit and implicit time integration methods. All these techniques

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form the basis of current physically based cloth simulation engines, and are suitable to be combined with hybrid approaches for real-time animation of clothing such as [CMT02, CSMT03].

6. The Virtual Dressmaker

In this section, we present a virtual tailor room which allows the interactive design and modification of clothing in a 3D Virtual Environment. In particular, we propose algorithms and interaction techniques for sewing and cutting garments during a physical cloth simulation, including the automatic modification of the underlying planar cloth patterns. The usability of the described methods is shown in a Virtual Reality application for interactive garment design that provides nearly live-size stereo projection combined with two-handed 6DOF interaction.

6.1. Introduction

Interactive garment design was a research topic in computer graphics already fourteen years ago, when geometric methods to model garment patterns as static three-dimensional surfaces on a workstation were proposed [HM90]. In the meantime, physical models and fast numerical algorithms now provide the simulation of the dynamic draping behaviour of clothing at interactive frame rates. Furthermore, Virtual Reality techniques and concepts, mostly offering input devices with six degrees of freedom and stereoscopic visualisation, have proved to be a useful alternative to traditional desktop applications with two-dimensional input devices and monoscopic displays in many areas.

While virtual prototyping in general has become a major area of research, the interactive design and simulation of clothing in Virtual Environments is still a relatively new topic. In [KSFS03, KSW*03, WKSS03, KFWS04] an approach to interactive cloth assembly and simulation in Virtual Reality is proposed. This application, called the *Virtual Dressmaker*, allows to construct and manipulate clothes coupled with physically based cloth simulation. Usability evaluations have shown that Virtual Environments can considerably improve the user performance in the assembly tasks compared to traditional desktop applications.

6.2. The Virtual Environment

Our Virtual Reality hardware setup consists of a desktop PC with a stereo table top display, a Virtual Table by Barco, with a visible screen of 140 x 105 cm^2 combined with LCD stereo shutter glasses for alternating projection (60 Hz). As input device we use an electromagnetic 6DOF tracker (Ascension, Flock of Birds) with three receivers. One receiver is attached to the shutter glasses and used to track the user's head position and to determine his gaze direction. The second receiver is attached to a transparent palette on which the interaction elements grouped in different sheets are back projected from the Virtual Table. The third receiver is attached to a pen with two buttons and is held in the dominant hand of the user. The virtual counterpart of the pen in the virtual scene is used to select, drag, and drop garment patterns, to operate the buttons and sliders on the palette, and

Now we want to proceed to suitable interaction techniques for cloth in order to provide manipulation tools for fitting garments and tailoring personalised clothes. We present an overview of a real time application for cloth simulation in virtual environments followed by special virtual tailor tools. to navigate in the scene. Hence, the application provides two handed interaction while allowing the user to exploit wellknown concepts of a desktop application like menus and sliders on the palette. Similar techniques for two handed interaction in VR have been explored and used extensively, e.g. in [AS95, BBMP97, SG97, CW99, BSP*03].

Our application is based on the software framework Studierstube API [SFGS96] developed at the University of Vienna. It provides a library extending the standard Open Inventor functionality to handle tracker events in order to embed desktop interaction elements within a VR framework. The application is coupled with a cloth simulation module to sew the prepositioned patterns together and drape the cloth over an avatar.

6.3. Interaction Elements

The Pen For assembling garments in the Virtual Dressmaker, the pen is used to freely manipulate the position of the garment patterns. By choosing a garment pattern, i.e. by pressing the front button on the pen, the object is selected, highlighted in orange, and is attached to the pen such that it can be moved in the space. To navigate the avatar, it is placed on a turntable which can be rotated around and translated along the vertical axis. Moreover, there is a possibility to zoom in and out of the scene and to move the camera position (viewing position). We therefore integrated the well known navigation methods Scene in Hand and Flying Vehicle [WO90] into the application which can be applied by pressing the pen's second button. Moreover, we improved the Scene in Hand tool by constraining the navigation possibilities to the degrees of freedom of the turntable. This results in a Constrained Scene in Hand navigation tool, which we think is the optimal choice among the given methods in the context of cloth assembly and design because the user is not prescinded from the working area and remains immersed in the scene.

Advanced Interactive Prepositioning In order to preposition cloth patterns that have to be bent during the sewing process, such as the sleeves of a pullover, we use the cloth simulation engine in combination with the input devices. To this end, Virtual Pins are integrated into the physical cloth simulation as constraints. Thus, we start the simulation for the sleeve only with the seams not yet connected. For this process we provide a menu on the palette to simulate the last touched pattern. Hence the user chooses the relevant pattern and starts the single pattern simulation. Then, we let the user pin parts of the cloth to arbitrary positions, while for the rest of the sleeve the according drape is simulated (Figure 15). When a satisfactory position is reached, the regular sewing and simulation process can be started.

The Palette Sheets The palette provides 3D widgets for buttons and sliders, resembling a 2D menu within the 3D Virtual Environment. Thus, we are able to display various virtual tools on the surface of the palette. Depending on the functions of the interface elements, they are grouped into categories. Each of these categories defines a *sheet*. In other words, a sheet is a set of interface elements with similar logical functionality. In order to group the interface elements in meaningful sets, we have implemented different sheets. There is a sheet to choose the avatar and the garment patterns from the database. Moreover, a navigation sheet is provided for precise positioning of the camera. For the simulation, a material sheet and a simulation sheet is implemented. Finally, for the tailoring process, the user can choose between a cutting and a sewing sheet.



Figure 16: The navigation and material sheets.

7. Constructing Clothes from Planar Patterns

In this section, we describe how three-dimensional virtual clothes, just as real clothes, can be constructed from planar patterns. This results in a natural starting condition for cloth animations, and recent work on cloth animation has focused on this approach [EWS96, EDC96, VMT00b, CK02, GFL03, EKS03]. The advantages of starting with planar cloth patterns are:

- Real clothes are also constructed from planar patterns. Thus, virtual garments can be designed like the real ones, e.g. by taking the patterns from a cloth manufacturer's CAD database.
- The triangulation of the planar patterns provide a natural parametrisation for the cloth simulation. For particle systems, the edges in the planar meshes can be taken as the spring's rest lengths [EWS96, VMT00b, CK02]. For finite element models, the discretised patterns allow the computation of the basis functions [EDC96, EKS03].
- Correct texture coordinates for rendering the simulated garments are obtained automatically by the planar rest state. They can be easily mapped to the three dimensional garment during the sewing process and provide a realistic visualisation.

A typical work flow within the Virtual Dressmaker starts by choosing a human model in the corresponding menu on the palette. This can be a body scan or some modelled 3D

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Figure 15: The sleeves of the pullover are prepositioned interactively. Image (a) shows the flat patterns around the 3D figure. In (b) the user wraps a sleeve around an arm using Virtual Pins.

figure. Then, a set of garment patterns can be chosen and positioned interactively around the avatar (see Figure 18). Both the interaction with the menus on the palette and the manipulation of the patterns in the scene is done with the pen. After positioning the patterns, the sewing and draping process can be started. During the physical cloth simulation, the user can still navigate freely in the scene and even pick and move parts of the garments. The design and development of virtual tools for the Virtual Dressmaker is independent of the cloth simulation engine which is decoupled from the Virtual Reality application via a client-server interface.

In the following, we first specify the necessary input data for our tailoring algorithms. Then, we describe the sewing process (i.e. the topological mergence of the patterns along seam lines) and the generation of a correct rest state for the physical cloth simulation.

7.1. Patterns and Seams

The input data for the sewing algorithm consists of planar patterns, seam information, and copies of the patterns positioned around the 3D figure. First, we need the twodimensional cloth patterns from which the garment shall be constructed. The single patterns are assumed to be arbitrary triangulated polygons. For an example, see the patterns of a woman's skirt in Figures 17, which are generated from real garment pattern shapes given by the boundary curve in a CAD system.

Second, the seam information has to be available. A seam belongs to exactly two (not necessarily distinct) patterns and consists of a starting and an end point in the respective patterns. We assume that the patterns are triangulated in such a way that two corresponding seam lines have the same number of vertices. More precisely, each seam is given by a list of pairs of vertices (u_i, v_i) with u_i and v_i being the corresponding vertices on two not necessarily distinct plane patterns (see Figure 17). Note that we exclude pathological cases like



Figure 17: The seam points of two triangulated planar patterns. The yellow and orange seams belong to the same pattern, whereas the green ones belong to two different patterns. Note that the red and blue seams on the very left respectively right hand side belong to borders on a third pattern of the skirt.

a vertex pair that belongs to a single triangle. Such a case can be handled by refining the given triangulation of the pattern locally. A new seam can be created by selecting them in pairs with the pen resulting in a seam list of vertex pairs. The seams are shown to the user by seam lines and for wrongly placed seams, the user can cancel the last defined seam by a button on the corresponding menu.

Finally, we need copies of the planar patterns that have been positioned to the approximately correct places around the virtual character, as seen in Figure 18. In order to provide a good initial position for the subsequent simulation, the requirements are:

- The patterns should not collide with the virtual character or with each other.
- The patterns should be as close to the virtual character as possible.

• It should be possible to connect the corresponding seams without penetrating the avatar.





Given these prepositioned patterns, the garment can be connected along the corresponding seam lines. Sewing the prepositioned cloth patterns together can be achieved by topologically merging the patterns along the defined seam lines. For all pairs of vertices (u_i, v_i) in all seams, the corresponding triangles are connected, and the seam points are moved halfway between the two original points.



Figure 19: The result of the simulation with correct texture coordinates along the seam line, where two patterns are connected.

8. Interactive Cutting

In this section, we present methods for the interactive cutting of clothes in 3D, including automatic modification of the planar patterns and the seam information. First, we describe how cuts can be defined on the garments, then we explain how they are realised on the 3D mesh and how they are transferred to the planar cloth patterns.

In order to define a cut on the simulated 3D mesh, the user has to mark points on the garment with the pen. Between

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Figure 20: A cut in a skirt is defined by the user.

these selected vertices, the mesh will be cut in straight lines by the cutting algorithm described in the next section.

Then, the corresponding patterns and seams are modified accordingly, and the physical simulation continues computing the drape and movement of the garment. An example is given in Figure 20.



Figure 21: The front pattern of the skirt before (left) and after (right) the cutting (cf. Figure 20). In the left pattern, the vertices selected by the user, and the edges that have to be cut are highlighted. Note that the cutting line in the patterns is not necessarily straight, as shown in the right image, due to the deformed state of the 3D mesh during the cutting.

For the underlying algorithm, there are two essentially different ways of modifying a mesh to model the effects of cutting. On the one hand, the affected triangles in the mesh can be split, on the other hand, the cuts can be modelled along the edges of the mesh. Since the second approach in general does not result in straight cuts, the vertices of the cut edges should be repositioned without affecting the underlying simulation. In both cases, the number of vertices in the mesh is increased by a cut, while the number of triangles remains the same only in the latter case. We choose to cut the meshes along the edges and to recalculate the position of the vertices because this approach fits well together with our sewing algorithm described in section 7. Thus, the special case of cutting along a seam line can be handled simply by removing the corresponding seams.

As mentioned above, in order to obtain a straight cutting line, we then move the vertices of the concerned edges in the 3D mesh towards the direction of the cutting plane, remaining in the plane spanned by the respective triangle (Figure 24). These changes of vertex positions in the 3D mesh are then transferred to the planar cloth patterns using barycentric coordinates. With this method, the relative displacements between the deformed state and the planar patterns are conserved, and a correct rest state for the continuation of the physical simulation is provided.

8.1. The Cutting Disk

The Cutting Disk provides an interactive cutting tool for executing plain cuts in the three-dimensional setting. It is most valuable for shortening garments like pairs of trousers or a shirt. Additionally it can be used to transform a dress into a skirt and a top as shown in Figure 23.



Figure 23: The use of the Cutting Disk: A cutting disk is placed at the waist of an avatar to cut a dress. A smooth cut is calculated by our algorithm and the dress is split into a top and a skirt which becomes apparent during the course of the simulation.

To define a cut the user can interactively position a thin cylinder, where the radius is adjustable. By the final cutting position, the user determines the plane where the garment mesh has to be cut. To determine the cutting line, we have to find a path on the edges of the mesh, where the corresponding triangles are cut by the cylinder. Therefore, we store all triangles affected by the Cutting Disk. The algorithm starts with the closest mesh vertex of the concerned triangles to the cutting plane. To find the cutting path, we choose the neighbouring vertex of the actual end of the cutting line which belongs to a cut triangle and which is the closest to the cutting plane. To avoid undesired closed loops (e.g. the vertices of one triangle), we keep track of the chosen vertices which can never be visited again except the start vertex. This vertex can only be reached when at most half of the affected triangles in the cutting list have been visited. The found vertices of the path are then projected onto the Cutting Disk. The

smoothing of the cut and the modification of the underlying cloth patterns is done as described in section 8.

The examples given in Figures 20 and 22 show some results of the proposed algorithms and interaction techniques for garment design. In Figure 22 a pullover is shortened, and the design of the collar is changed. All simulation images are directly captured from our Virtual Dressmaker application while accomplishing the given task.



Figure 24: A pullover is shortened by cutting a piece off. The first image shows the cut without aligning the vertices along the cut edges, in the second the vertex positions have been corrected and a straight cut is obtained. All the modifications are transferred to the planar patterns.

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Figure 22: *Image (a) shows the simulated pullover, together with the skirt. In (b) the user has shortened the pullover by cutting off a broad band on the lower part. Note that the cutting lines in this example cross two seams. In image (c) the design of the collar has been modified interactively.*

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Tutorial

Part 2: Techniques for Virtual Clothing

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Abstract

Despite numerous methods available for cloth simulation, virtual garment prototyping has yet to find its way toward the garment industry, the main issues being simulation accuracy and the potentiality for reproducing the complex behavior of complex garment models. These goals can only be reached through an optimal combination of modeling techniques and numerical methods that combines high computation efficiency with the versatility required for simulating intricate garment designs. We here describe optimal choices illustrated by their integration in a design and simulation tool that allow interactive prototyping of garments along drape motion and comfortability tests on animated postures.



1. Introduction

Garment simulation for prototyping application still remains a challenge, because of the high complexity of the simulation context, which goes far beyond the current application fields of the available cloth simulation systems.

Facing the high requirements of garment designers toward not only realism but also quantitative mechanical accuracy on draping and motion, the complexity and diversity of real garment models makes it a real challenge to design a garment simulation system [12]. Among the main difficulties are:

* The size of the garments, which can be meter-long and yet need to be simulated at millimeter accuracy.

* The intricate and highly variable shape of the garments, which interacts through complex contact patterns with the body (which is itself a complex deformable entity), as well as other garments.

* The highly deformable nature of cloth, which translate very subtle mechanical variations into large draping and motion variations which modify completely the visual appearance of garment models.

* The highly intricate anisotropic and nonlinear mechanical behavior of garments, requiring accurate measurement, modeling, and complex numerical methods for their resolution.

Mechanical simulation of cloth is a topic that is currently well explored, and many systems are already able to capture and simulate the mechanical properties of cloth. Among the best known techniques, spring-mass-like particle systems [4] take a large share, because these methods are fast and versatile, but unfortunately quite inaccurate when it comes to modeling complex anisotropic and nonlinear mechanical behaviors. On the other side, accurate methods such as finite elements [6] need huge computational requirements, and are not suited for the simulation of complete garments.

Among existing tools aimed at garment prototyping, most of them are focused only on draping, and are not suited for delivering accurate simulations of garments on animated characters. Animation is however a key issue in garment prototyping, as the motion of garments accounts a lot for the final visual look-and-feel of a dressing style. This however requires the simulation to reach a higher level of accuracy, with simulation of viscous behavior of cloth materials along with numerical integration methods that preserve the actual motion of the cloth surfaces along time. While many cloth simulation applications are available as commercial products, some of them capable of dealing with complete garments, none offers the accuracy necessary for an actual garment prototyping application, and therefore none is currently used in the garment industry.

We here intend to present how to fulfill the requirements of a garment designer, integrating state-of-the-art methods which combine both requirements of mechanical accuracy with the power and versatility required for simulating complete dressing styles on animated characters. Among the main features available for the garment designer are pattern design with interactive garment fitting evaluation, high-quality animation previews on moving characters, along with the possibility of managing dressing styles composed by several complex multilayer garments with many different materials and seamings [16]. We are putting emphasis on the following aspects:

* In Section 2, we describe a general mechanical model for cloth, which combines the versatility of particle systems with the accuracy of surface-based models, able to simulate the complex anisotropic nonlinear viscoelastic behaviors required for the accurate reproduction of the behavior of cloth, not only for draping applications but also for dynamic simulation of garment motion on animated characters.

* In Section 3, we discuss the particular problem of numerical integration, putting particular emphasis on studying the suitability of the various techniques to dynamical simulation of moving cloth through evaluation of their numerical damping.

* Several additional techniques required for simulating complex garments are also discussed, such as collision detection and processing techniques in Section 4.

* Finally, we show in Section 5 the results and the potentialities of all these methods put together in a system which is actually used to design and simulate complex virtual garments models.



Figure 1: Complex garments are still challenging to animate.

2. Simulating the Mechanics of Cloth

2.1. Mechanical Properties of Cloth

2.1.1. Description

The mechanical behavior of fabric is inherent in the nature and molecular structure of the fiber material constituting the cloth, and as well the way these fibers are arranged in the fabric structure. Fabric fibers can be organized in several ways in the cloth surface. The main structures are: * **Woven Fabrics**: Threads are orthogonally aligned and interlaced alternately using different patterns (such as plain or twirl).

* Knitted fabrics: Threads are curled along a given pattern, and the curls are interlaced on successive rows.

* Non-woven fabrics: There are no threads, and the fibers are arranged in an unstructured way, such as paper fibers.

Woven fabrics are the most commonly used in garments. They are relatively stiff though thin, easy to produce, and may be used in a variety of ways in many kind of design. In contrast, knitted fabrics are loose and very elastic. They are usually employed in woolens or in underwear. This structure greatly influences the mechanical behavior of the fabric material, which is mainly determined by:

* The nature of the fiber: Wool, cotton, synthetic,...

* The thread structure: Diameter, internal fiber and yarn structure,...

* The thread arrangement: Woven or knitted, and particular pattern variation.

* The pattern properties: Tight or loose.

These properties are critical to the stiffness of the material, its ability to bend, and its visual appearance.



Figure 2: Different woven fabric patterns: Plain, Twirl, Basket, Satin.

The mechanical properties of deformable surfaces can be grouped into four main families:

* **Elasticity**, which characterizes the internal forces resulting from a given geometrical deformation.

* **Viscosity**, which includes the internal forces resulting from a given deformation speed.

* **Plasticity**, which describes how the properties evolve according to the deformation history.

* **Resilience**, which defines the limits at which the structure will break.

Most important are the elastic properties that are the main contributor of mechanical effects in the usual contexts where cloth objects are used. Deformations are often small and slow enough to make the effect of viscosity, plasticity and resilience insignificant. One major hypothesis is that **quasistatic models** in the domain of elastic deformations will suffice for models intended to simulate the rest position of the garment on an immobile mannequin (draping). However, when a realistic animation is needed, the parameters relating energy dissipation through the evolution of the deformation are also needed, and complete **dynamic models** including viscosity and plasticity should be used.

Depending on the amplitude of the mechanical phenomena under study, the curves expressing mechanical properties exhibit shapes of varying complexity. If the amplitude is small enough, these shapes may be approximated by straight lines. This **linearity** hypothesis is a common way to simplify the characterization and modeling of mechanical phenomena.

It is common in elasticity theory to consider that the orientation of the material has no effect on its mechanical properties (**isotropy**). This however is inappropriate for cloth, as its properties depend considerably on their orientation relative to the fabric thread.

Elastic effects can be divided into several contributions:

* Metric elasticity, deformations along the surface plane.

* **Bending elasticity**, deformations orthogonally to the surface plane.

Metric elasticity is the most important and best studied aspect of fabric elasticity. It is usually described in terms of strain-stress relations. For linear elasticity, the main laws relating the strain e to the stress s involve three parameters, which are:

* The Young modulus E, summarizing the material's reaction along the deformation direction.

* **The Poisson coefficient v**, characterizing the material's reaction orthogonal to the deformation direction.

* The Rigidity modulus G, pertaining to oblique reactions.

Along the two orthogonal directions i and j, these relations, named Hook's Law, Poisson Law and Simple Shear Law relating the stress ε to the strain σ are respectively expressed as follows:

$$\varepsilon_{ii} = \frac{1}{E_i} \sigma_{ii} \qquad \varepsilon_{jj} = \frac{V_{ij}}{E_i} \sigma_{ii} \qquad \varepsilon_{ij} = \frac{1}{G_{ij}} \sigma_{ij} \qquad (1)$$

Cloth materials are two-dimensional surfaces for which two-dimensional variants of the elasticity laws are suitable. They are not isotropic, but the two orthogonal directions defined by the thread orientations can be considered as the main orientations for any deformation properties. In these **orthorombic** cloth surfaces, the two directions are called **weft (u)** and **warp (v)**, and they have specific Young modulus and Poisson coefficients, $\mathbf{E}_{\mathbf{u}}$, $\mathbf{v}_{\mathbf{u}}$ and $\mathbf{E}_{\mathbf{v}}$, $\mathbf{v}_{\mathbf{v}}$ respectively. The elasticity law can be rewritten in terms of these directions as follows:

$$\begin{bmatrix} \sigma_{uu} \\ \sigma_{vv} \\ \sigma_{uv} \end{bmatrix} = \frac{1}{1 - v_u v_v} \begin{bmatrix} E_u & v_v E_u & 0 \\ v_u E_v & E_v & 0 \\ 0 & 0 & G(1 - v_u v_v) \end{bmatrix} \begin{bmatrix} \varepsilon_{uu} \\ \varepsilon_{vv} \\ \varepsilon_{uv} \end{bmatrix}_{(2)}$$

Energetic considerations imply the above matrix to be symmetric, and therefore the products $\mathbf{E}_{u} \mathbf{v}_{v}$ and $\mathbf{E}_{v} \mathbf{v}_{u}$ are equal. Considering isotropic materials, we also have the following relations:

$$E_{u} = E_{v} v_{u} = v_{v} G = \frac{E}{2(1-v)}$$
(3)

A similar formulation can be obtained for bending elasticity. However the equivalent of the Poisson coefficient for bending is usually taken as null. The relation between the curvature strain τ and stress γ is expressed using the flexion modulus **B** and the flexion rigidity **K** (often taken as null) as follows:

$$\begin{bmatrix} \tau_{uu} \\ \tau_{vv} \\ \tau_{uv} \end{bmatrix} = \begin{bmatrix} B_u & 0 & 0 \\ 0 & B_v & 0 \\ 0 & 0 & K \end{bmatrix} \begin{bmatrix} \gamma_{uu} \\ \gamma_{vv} \\ \gamma_{uv} \end{bmatrix}$$
(4)

While elasticity expresses the relation between the force and the deformation, viscosity expresses the relation between the force and the deformation speed in a very similar manner. To any of the elasticity parameters can be defined a corresponding viscosity parameter obtained by substitution of the stresses $\boldsymbol{\varepsilon}$ and $\boldsymbol{\gamma}$ by their derivatives along time $\boldsymbol{\varepsilon}'$ and $\boldsymbol{\gamma}'$.

While the described linear laws are valid for small deformations of the cloth, large deformations usually enter the nonlinear behavior of cloth, where there is no more proportionality between strain and stress. This is practically observed by observing a "limit" in the cloth deformation as the forces increases, often preceding rupture (resilience), or remnant deformations observed as the constraints are released (plasticity). A common way to deal with such nonlinear models is to assume weft and warp deformation modes as still being independent, and replace each linear parameter E_u , E_v , G, B_u , B_v by nonlinear strain-stress behavior curves.

2.1.2. Measuring the Mechanical Properties of Cloth

The garment industry needs the measurement of major fabric mechanical properties through normalized procedures that guarantee consistent information exchange between garment industry and cloth manufacturers. The **Kawabata Evaluation System for Fabric** (KES) is a reference methodology for the experimental observation of the elastic properties of the fabric material. Using five experiments, fifteen curves are obtained, which then allow the determination of twenty-one parameters for the fabric, among them all the linear elastic parameters described above, except for the Poisson coefficient.

Five standard tests are part of KES for determining the mechanical properties of cloth, using normalized measurement equipment. The **tensile test** measures the force/deformation curve of extension for a piece of fabric of normalized size along weft and warp directions and allows the measurement of E_u and E_v along with other parameters assessing nonlinearity and hysteresis. The **shearing test** is the same experiment using shear deformations, which

allows the measurement of **G**. The **bending test** measures the curves for bending deformation in a similar way, and allows the measurement of B_u and B_v . Finally, the **compression test** and the **friction test** allow the measurement of parameters related to the compressibility and the friction coefficients.



Figure 3: Measuring cloth properties using KES.

While the KES measurements allow determination of parameters assessing the nonlinearity of the behavior curves and some evaluation of the plasticity, other methodologies, such as the FAST method, use simpler procedures to determine the linear parameters only.

While the Kawabata measurements and similar systems summarize the basic mechanical behaviors of fabric material, the visual deformations of cloth, such as buckling and wrinkling, are a complex combination of these parameters with other subtle behaviors that cannot be characterized and measured directly.

In order to take these effects into account, other tests focus on more complex deformations. Among them, the draping test considers a cloth disk of given diameter draped onto a smaller horizontal disc surface. The edge of the cloth will fall around the support, and produce wrinkling. The wrinkle pattern can be measured (number and depth of the wrinkles) and used as a validation test for simulation models.

Tests have also been devised for measuring other complex deformation of fabric material, mostly related to bending, creasing and wrinkling.

2.2. Cloth Simulation Systems

Cloth being approximated as a thin surface, its mechanical behavior is decomposed in in-plane deformations (the 2D deformations along the cloth surface plane) and bending deformation (the 3D surface curvature).

The in-plane behavior of cloth is described by relationships relating, for any cloth element, the stress σ to the strain ε (for elasticity) and its speed ε' (for viscosity) according the laws of viscoelasticity. For cloth materials, strain and stress are described relatively to the weave directions weft and warp following three components: weft and warp elongation (**uu** and **vv**), and shear (**uv**). Thus, the general viscoelastic behavior of a cloth element is described by strain-stress relationships as follows:

$$\sigma_{uu}(\varepsilon_{uu},\varepsilon_{vv},\varepsilon_{uv},\varepsilon_{uu}',\varepsilon_{v'}',\varepsilon_{uv}')$$

$$\sigma_{vv}(\varepsilon_{uu},\varepsilon_{vv},\varepsilon_{uv},\varepsilon_{uu}',\varepsilon_{v'}',\varepsilon_{uv}')$$

$$\sigma_{uv}(\varepsilon_{uu},\varepsilon_{vv},\varepsilon_{uv},\varepsilon_{uu}',\varepsilon_{v'}',\varepsilon_{uv}')$$
(5)

Assuming to deal with an orthotropic material (usually resulting from the symmetry of the cloth weave structure relatively to the weave directions), there is no dependency between the elongation components (uu and vv) and the shear component (uv). Assuming null Poisson coefficient as well (a rough approximation), all components are independent, and the fabric elasticity is simply described by three independent elastic strain-stress curves (weft, warp, shear), along with their possible viscosity counterparts.

In the same manner, viscoelastic strain-stress relationships relate the bending momentum to the surface curvature for weft, warp and shear. With the typical approximations used with cloth materials, the elastic laws are only two independent curves along weft and warp directions (shear is neglected), with their possible viscosity counterparts.

The issue is now to define a model for representing these mechanical properties on geometrical surfaces representing the cloth. These curved surfaces are typically represented by polygonal meshes, being either triangular or quadrangular, and regular or irregular.

Continuum mechanics are one of the schemes used for accurate representation of the cloth mechanics. Mechanical equations are expressed along the curved surface, and then discretized for their numerical resolution. Such accurate schemes are however slow and not sufficiently versatile for handling large deformations and complex geometrical constraints (collisions) properly. Finite Element methods express the mechanical equations according to the deformation state the surface within well-defined elements (usually triangular or quadrangular). Their resolution also involves large computational charges. Another option is to construct a model based on the interaction of neighboring discrete points of the surface. Such particle systems allow the implementation of simple and versatile models adapted for efficient computation of highly deformable objects such as cloth.

2.2.1. Spring-Mass Models

The simplest particle system one can think of is spring-mass systems. In this scheme, the only interactions are forces exerted between neighboring particle couples, similarly as if they were attached by springs (described by an force/elongation law along its direction, which is actually a rigidity coefficient and a rest length in the case of linear springs). Spring-mass schemes are very popular methods, as they allow simple implementation and fast simulation of cloth objects. There has also been recent interest in this method as it allows quite a simple computation of the Jacobian of the spring forces, which is needed for implementing semi-implicit integration methods (see Section 3).

The simplest approach is to construct the springs along the edges of a triangular mesh describing the surface. This however leads to a very inaccurate model that cannot model accurately the anisotropic strain-stress behavior of the cloth material, and also not the bending. More accurate models are constructed on regular square particle grids describing the surface. While elongation stiffness is modeled by springs along the edges of the grid, shear stiffness is modeled by diagonal springs and bending stiffness is modeled by leapfrog spring along the edges. This model is still fairly inaccurate because of the unavoidable crossdependencies between the various deformation modes relatively to the corresponding springs. It is also inappropriate for nonlinear elastic models and large deformations. More accurate variations of the model consider angular springs rather than straight springs for representing shear and bending stiffness, but the simplicity of the original spring-mass scheme is then lost.



Figure 3: Using length or angle springs for simulating cloth with a square particle system grid.

2.2.2. An Accurate Particle System Model

Because of the real need of representing accurately the anisotropic nonlinear mechanical behavior of cloth in garment prototyping applications, spring-mass models are inadequate, and we need to find out a scheme that really simulates the viscoelastic behavior of actual surfaces. For this, we have defined a particle system model that relates this accurately over any arbitrary cloth triangle through simultaneous interaction between the three particles which are the triangle vertices. Such a model integrates directly and accurately the strain-stress model defined in Part 2.1 using polynomial spline approximations of the strain-stress curves, and remains accurate for large deformations.

In this model, a triangle element of cloth is described by 3 2D coordinates (**ua**, **va**), (**ub**, **vb**), (**uc**, **vc**) describing the location of its vertices **A**, **B**, **C** on the weft-warp coordinate system defined by the directions **U** and **V** with an arbitrary origin. They are orthonormal on the undeformed cloth (Fig.4). Out of them, a precomputation process evaluates the following values:

$$Rua = d^{-1} (vb - vc) Rva = -d^{-1} (ub - uc)
Rub = -d^{-1} (va - vc) Rvb = d^{-1} (ua - uc)
Ruc = d^{-1} (va - vb) Rvc = -d^{-1} (ua - ub)
with
d = ua (vb - vc) + ub (vc - va) + uc (va - vb) (6)$$

During the computation process, the current deformation state of the cloth triangle is evaluated using the current 3D direction and length of the deformed weft and warp direction vectors U and V. They are computed from the current positions Pa, Pb, Pc of its supporting vertices as follows:

$$U = Rua Pa + Rub Pb + Ruc Pc$$

$$V = Rva Pa + Rvb Pb + Rvc Pc$$
(7)

The current in-plane strains $\boldsymbol{\epsilon}$ of the cloth triangle is then computed with the following formula:

$$\varepsilon_{uv} = \frac{|U| - 1}{\sqrt{2}} \quad \varepsilon_{vv} = |V| - 1 \tag{8}$$
$$\varepsilon_{uv} = \frac{|U+V|}{\sqrt{2}} - \frac{|U-V|}{\sqrt{2}}$$

We have chosen to replace the traditional shear deformation evaluation based on the angle measurement between the thread directions by an evaluation based on the length of the diagonal directions. The main advantage of this is a better accuracy for large deformations (the computation of the behavior of an isotropic material under large deformations remains more axis-independent).



Figure 4: A triangle of cloth element defined on the 2D cloth surface (left) is deformed in 3D space (right) and its deformation state is computed from the deformation of its weft-warp coordinate system.

For applications that model internal in-plane viscosity of the material, the "evolution speeds" of the weave direction vectors are needed as well. They are computed from the current triangle vertex speeds **Pa'**, **Pb'**, **Pc'** as follows:

$$U' = Rua Pa' + Rub Pb' + Ruc Pc'$$

$$V' = Rva Pa' + Rvb Pb' + Rvc Pc'$$
(9)

Then, the current in-plane strain speeds $\boldsymbol{\epsilon}'$ of the triangle is computed:

$$\varepsilon_{uu'} = \frac{U \cdot U'}{|U|} \qquad \varepsilon_{vv'} = \frac{V \cdot V'}{|V|} \tag{10}$$
$$\varepsilon_{uv'} = \frac{(U+V) \cdot (U'+V')}{|U+V|\sqrt{2}} - \frac{(U-V) \cdot (U'-V')}{|U-V|\sqrt{2}}$$

At this point, the in-plane mechanical behavior of the material can be expressed for computing the stresses σ out of the strains ϵ (elasticity) and the strain speeds ϵ' (viscosity) using the curves discussed in Part 2.2. Finally, the force contributions of the cloth triangle to its support vertices computed from the stresses σ as follows:

$$Fa = -\frac{d}{2} \left(\left(Rua \ \sigma_{uu} + Rva \ \sigma_{uv} \right) \frac{U}{|U|} + \left(Rua \ \sigma_{uv} + Rva \ \sigma_{vv} \right) \frac{V}{|V|} \right) \quad (11)$$

$$Fb = -\frac{d}{2} \left(\left(Rub \ \sigma_{uu} + Rvb \ \sigma_{uv} \right) \frac{U}{|U|} + \left(Rub \ \sigma_{uv} + Rvb \ \sigma_{vv} \right) \frac{V}{|V|} \right)$$

$$Fc = -\frac{d}{2} \left(\left(Ruc \ \sigma_{uu} + Rvc \ \sigma_{uv} \right) \frac{U}{|V|} + \left(Ruc \ \sigma_{uv} + Rvc \ \sigma_{vv} \right) \frac{V}{|V|} \right)$$

It is important to note that when using semi-implicit integration schemes (see Section 3), the contribution of these forces in the Jacobian $\partial F/\partial P$ and $\partial F/\partial P'$ can easily be computed out of the curve derivatives $\partial \sigma/\partial \varepsilon$ and the orientation of the vectors U and V.

The model has been extended to handle anisotropic curvature stiffness through bending momentums applied along edges, for which the angle between the adjacent elements give an evaluation of the local surface curvature along the orthogonal direction. Additional mechanical features are integrated in the model as well for simulating the behavior of complex garment features (Fig.6).



Figure 6: Drape accuracy between a simple spring-mass system along the edges of the triangle mesh (left) and the proposed accurate particle system model (center). Color scale shows deformation. The spring-mass model exhibits inaccurate local deformations, along with an excessive "Poisson" behavior. This is not the case with the accurate model, which may still model the "Poisson" effect if needed (right, with a Poisson coefficient 0.5). The spring-mass model is also unable to simulate anisotropic or nonlinear

models accurately.



Figure 7: The proposed model simulates accurately anisotropic bending stiffness, with possible rest curvature defined on the surface (left). Rest curvature may also be defined along precise lines (center). Lines may also carry additional stiffness with their own custom rest length (right). All these features bring lot of potentialities for designing complex garment models.

3. Numerical Integration

The equations resulting from the mechanical formulation of particle systems do usually express particle forces **F** depending on the state of the system (particle positions **P** and speeds **P'**). In turn, particle accelerations **P''** is related to particle forces **F** and masses **M** by Newton's 2nd law of dynamics. This leads to a second-order ordinary differential equation system, which is turned to first-order by concatenation of particle position **P** and speed **P'** into a state vector **Q**. A vast range of numerical methods has been studied for solving this kind of equations.

We have conducted extensive tests for benchmarking numerous integration methods, using performance, accuracy, stability and robustness as criteria. We have selected three candidates, each of which performs best in its own context:

- * *1st-order semi-implicit Backward Euler*, which seems to be the best robust general-purpose method for any relaxation task (garment assembly and draping) [1] [14].
- * 2nd-order semi-implicit Backward Differential Formula, which offers increased dynamic accuracy along time (garment simulation on animated characters), at the expense of robustness (unsuited for draping during interactive design) [7].
- * 5th-order explicit Runge-Kutta with timestep control, which offers very high non-dissipative dynamic accuracy (accurate simulation of viscous and dissipative parameters in animated garments), at the expense of computation time (requires small time steps depending on the numerical stiffness, unsuited for stiff materials and refined discretizations) [4].

Our implementation integrates these three methods, and dynamically switches between them depending on the simulation context.

3.1. Discussing Integration Methods

3.1.1. Implicit Integration Methods

The most widely-used method for cloth simulation is currently the semi-implicit Backward Euler method, which was first used by Baraff et al [1] in the context of cloth simulation. As any implicit method, it alleviates the need of high accuracy for the simulation of stiff differential equations, offering convergence for large timesteps rather than numerical instability (a step of the semi-implicit Euler method with "infinite" timestep is actually equivalent to an iteration of the Newton resolution method) [15].

The formulation of a generalized implicit Euler integration is the following:

$$Q_{(t+dt)} - Q_{(t)} = Q'_{(t+\alpha dt)} dt$$
 (12)

The derivative value is not known at a moment after \mathbf{t} , and is then extrapolated from the value at moment \mathbf{t} using the Jacobian, leading to the semi-implicit expression which requires the resolution of a linear system:

$$Q_{(t+dt)} - Q_{(t)} = \left(I - \alpha \frac{\partial Q'}{\partial Q_{(t)}} dt\right)^{-1} Q'_{(t)} dt \qquad (13)$$

We have introduced the coefficient α so as to modulate the "impliciticy" of the formula. Hence, $\alpha = 1$ is the regular implicit Backward Euler step (stable), whereas $\alpha = 0$ is the explicit Forward Euler step (unstable), and $\alpha = 1/2$ is the 2nd-order implicit Midpoint step (most accurate, at the threshold of stability).

The α parameter is a good handle for adjusting the compromise between stability and accuracy. While maximum robustness is obviously observed for large values, reducing its value increases accuracy (reduces numerical damping) at the expense of stability, and speeds up the computation as well (better conditioning of the linear system to be resolved).

Better accuracy can also be obtained through the use of the 2nd-order Backward Differential Formula (BDF-2), as described by Hauth et al [7]. This uses the previous state of the system for enhancing accuracy up to 2nd-order, with a minimal impact on the computation charge. Its generalized implicit expression is:

$$Q_{(t+dt)} - Q_{(t)} = \beta \left(Q_{(t)} - Q_{(t-dt)} \right) + Q'_{(t+\alpha dt)} \, \delta t$$

with
$$\beta = \frac{2 \, \alpha - 1}{2 \, \alpha + 1} \qquad \text{and} \qquad \delta t = \frac{2}{2 \, \alpha + 1} \, dt \qquad (14)$$

And its semi-implicit expression is:

$$Q_{(t+dt)} - Q_{(t)} = \left(I - \alpha \frac{\partial Q'}{\partial Q_{(t)}} \delta t\right)^{-1} \left(\beta \left(Q_{(t)} - Q_{(t-dt)}\right) + Q'_{(t)} \delta t\right)^{(15)}$$

While $\alpha = 1$ is the regular implicit BDF-2 step, $\alpha = 0$ is the explicit Leapfrog method, and $\alpha = 1/2$ is again the implicit Midpoint method. Best accuracy is offered for $\alpha = 1/\sqrt{3}$, where the method is 3rd-order (moderately stable).

Compared to Backward Euler, the main interest of the BDF-2 method is that it exhibits better accuracy for dynamic simulation over time (less numerical damping) for moderately stiff numerical contexts (at the expense of reduced robustness for nonlinear situations). For very stiff contexts however, this benefit disappears. While it is possible to implement higher-order BDF methods, their interest is reduced by their lack of stability, and high accuracy could be more efficiently reached using high-order explicit methods. Stability of implicit methods is also affected by the nonlinearities of the mechanical model.



Figure 8: Stability test: A square of cloth is initially deformed with large random perturbations, and then simulated using various timesteps.

3.1.2. Explicit Integration Methods

Unlike implicit methods, explicit methods do not offer convergence to equilibrium if the timestep is too large compared to the numerical stiffness of the equations. On the other hand, they are very simple to implement, and much compute much faster than their implicit counterpart for reaching a given accuracy. This is particularly true for highorder methods, which offer very high accuracy if the timestep is small enough, but diverge abruptly if it exceeds a threshold (related to the stiffness of the equations). This is why an efficient timestep control scheme is essential for the implementation of these methods.

While the explicit 1st-order Euler and 2nd-order Midpoint methods should be restricted to simple applications (beside their simplicity, they have no benefits compared to their implicit counterparts), a popular choice is the 5th-order Runge-Kutta scheme with embedded error evaluation [11]. It is a six-stage iteration process where the computed error magnitude can be used for controlling the adequate timestep very accurately, depending on accuracy and stability requirements. Unlike implicit methods, this method yields a very good guaranteed accuracy (resulting from the highorder, but which may require very small timesteps), which is particularly important for problems where energy conservation is a key issue (for example, evaluating the effect of viscous parameters in the motion of fabrics) (Fig.8). On the other hand, explicit methods are quite unsuited for the fast relaxation of the static cloth draping applications.



Figure 9: Evaluation of numerical damping of various integration methods using energy dissipation plots along time(50cm x 50cm square cloth, initially horizontal, attached along one edge, linear isotropic 100N/m, 100g/m2, 2cm2 elements, no dissipative parameters). 5thorder Runge-Kutta (up) accurately preserves the total energy along time, a good amount of it being transferred to elastic energy through small-scale mesh jittering (timesteps between 0.0001s and 0.00001s). Implicit methods such as Inverse Euler (down) damp small-scale motion.



Figure 10: The 3rd-order BDF2 variation (down) preserves energy significantly better than Inverse Euler (up).

3.2. Implementation Issues

While there are no particular issues related to the implementation of explicit integration methods, semiimplicit methods require the resolution of large sparse linear equations systems, which are mainly constructed from the Jacobian of the mechanical law $\partial F/\partial P$ and $\partial F/\partial P'$ (their sparse structure relates the mechanical dependency between the particles). Among possible speed-up approximations, the Implicit-Explicit method described in [5] neglects the Jacobian terms generated by the non-stiff forces (which are then explicitly integrated).

A choice candidate for resolving this linear system is the Conjugate Gradient method, which is iterative and thus offers compromise between computation charge and symmetric accuracy, and which also allows efficient implementation for sparse systems.

Among possible optimizations are linearization schemes aimed at performing the computation using a constant approximation of the Jacobian, so as to implement preprocessing optimizations in the resolution. While giving reasonable benefits for draping applications, these approximations however generate large "numerical damping" that slow down convergence and alter highly the motion of the cloth along time [2] [3] [8] [10].

The only solution for simulating the accurate motion of cloth was indeed to use real value of the Jacobian corresponding to the current state of the system. We have taken advantage of the Conjugate Gradient method which only needs the Jacobian matrix products with given vectors to compute these products "on the fly" directly from the system state, skipping the sparse explicit storage of the matrix for each frame. Our system actually allows performing partial linearization of the Jacobian, so as to use the linearization ratio offering the best tradeoff between motion accuracy and stability, depending on the simulation context.

4. Collision Processing

Collision detection is indeed one of the most timeconsuming tasks when it comes to simulate virtual characters wearing complete garments [9]. This task is performed through an adapted bounding-volume hierarchy algorithm, which uses a constant Discrete-Orientation-Polytope hierarchy constructed on the mesh, and optimization for self-collision detection using curvature evaluation on the surface hierarchy. This algorithm is fast enough for allowing full collision and self-collision detection between all objects of the scene with acceptable impact on the processing time (rarely exceeds 20% of the total time). Thus, body and cloth meshes are handled totally symetrically by the collision detection process, ensuring perfect versatility of the collision handling between the body and the several layers of garments [13].

Collision response is handled using a geometrical scheme based on correction of mesh position, speed and acceleration. This scheme ensures good accuracy and stability without the need of large nonlinear forces that alter the numerical resolution of the mechanical model. Our model simulates contact forces through a perfectly damped reaction model, associated to a Coulombian (solid) friction model.

The implemented collision model ensures full mesh-tomesh collision response, which can deal with very complex multilayer collisions configurations involving several surfaces. The collision processing is therefore general enough for handling contacts between the several garments of a complex dress style, as well as the interactions between complex fold patterns when animating ample gestures. The model is also accurate enough for reproducing accurately friction behavior, allowing for example pants to hold to the waist with friction alone during character motion, without "cheating" using geometrical attachments. Good stability allows the simulation of complete multilayer garments with millimeter collision thickness despite large cloth speed and tension produced by complex character motion.



Figure 11: Efficient and robust collision processing is important for simulating complex cloth.

5. An Interactive System for Garment Prototyping

While essential, computational techniques alone are not sufficient for producing a powerful tool allowing accurate and convenient creation and prototyping of complex garments. We have integrated all these techniques into a garment design and simulation tool aimed at prototyping and virtual visualization, and allowing fashion designers to experiment virtually new collections with high-quality preview animations, as well as pattern makers to adjust precisely the shape and measurements of the patterns to fit the body optimally for best comfort.



Fig.12: A garment design interface showing both 2D pattern shapes and 3D garment drape.

The high level of interactivity required by these features necessitates simultaneous computation of the 3D garment updated immediately to each design modification done to the patterns. Our design and simulation tool provides a dual view of the garment, featuring both the 2D view of the pattern shapes cut on the fabric and the 3D view of the garment worn by a virtual character, with tight synchronization (Fig.12). Any editing task carried out in one view is directly displayed in the other view.



Figure 13: Between real and virtual: Our system offers high-quality garment simulation, along with highly interactive pattern 2D-3D design and preview tools allowing complex garment models to be designed efficiently with many features such as seams, buttons, pockets, belts...

5.1. Interactive Garment Editing Tools

The system features a fast Constrained Delaunay triangulation scheme that allows the discretization of complex patterns described as polygonal lines of control points (2D locations on the fabric). The system allows variable discretization densities over the mesh, as well as size anisotropy (elements elongated in a given direction), for representing adaptively complete garments from large surfaces to intricate details. The interactivity of the system is based on two main features (Fig.12):

* Mesh mapping update: The 2D displacement of any control point of the pattern shape on the cloth surface immediately updates the mesh of that pattern on the cloth, while leaving the 3D drape position of the cloth constant. For obtaining this, each vertex of the mesh keeps track of a weighted sum of the pattern control points, which is computed during the triangulation process. This allows any measurement or shape editing to be directly taken into account by the mechanical simulation without any heavy recomputation, for immediate feedback of any pattern sizing adjustment.

* *Mesh topology reconstruction*: When the topology of the pattern mesh is changed (rediscretization, new features, etc), the 3D drape position of the new mesh is automatically recomputed from the drape position of the old one. During

this process, advanced algorithms compute, for each mesh vertex of the new mesh, the location of the surface of the old mesh having identical 2D coordinates on the fabric. Extrapolation methods are used for computing the location of vertices which are located outside the old surface. This allows pattern design changes (new features, darts, seams...) to be added and modified without needing re-assembling and re-draping the garment on the virtual body.



Figure 14: Designing complex garments

5.2. Interactive Garment Prototyping

Put together, these techniques greatly enhance the workflow of garment prototyping. For instance, an initial garment could be quickly draped over a character using a rough mesh. Then, the designer could enhance the pattern shapes, while mesh mapping update automatically alter the mechanical state of the draped garment, changing the draping shape. Once the garment design is ready, a highaccuracy drape is automatically produced using topology reconstruction with a refined mesh.

Combined with the accuracy and speed of the proposed mechanical simulation engine, tasks such as comfortability evaluations are open to the garment designer, through the addition of several visualization tools, such as (Fig. 13):

* Preview of fabric deformations and tensions along any weave orientation.

* Preview of pressure forces of the garment on the body skin.

* Immediate update of these evaluations according to pattern reshaping and sizing, fabric material change, and body measurements and posture change.

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Figure 12: Interactive garment design: From an initial draped pattern (A), editing the 2D contour of a pattern while keeping its 3D shape constant (B) allows mechanical simulation to smoothly drape the pattern to its new 3D shape (C). If pattern editing involves topology or seaming changes (D), the pattern surface is automatically reconstructed in the same position (E) using interpolation if necessary, and simulation performs the draping (F). The surface may be refined at any time (G) for more accuracy.



Figure 13: Interactive garment prototyping and comfort evaluation: More than the garment shape and appearance design (left), the system also allows mechanical comfortability data to be evaluated directly with any change of the pattern design and sizing (right).

Dynamic surface remeshing allows the best compromise between accuracy and computation speed to be selected adaptively according to the needs of the garment designer. For instance, while the garment assembly process can be carried out in a matter of seconds using an approximate mechanical model on a rough garment surface mesh, the garment designer may then switch to a more accurate model for tasks such as accurate draping and comfort evaluation. The model is still efficient enough to react interactively to design changes with garments made of ten thousands polygons, an accurate draping being obtained in a few minutes. Practical geometric accuracy is roughly limited by using 5 millimeters elements (Fig.14). Using time-accurate computation on animated characters, a high-quality catwalk is computed in a matter of a few hours.

5.3. Perspectives

Interactive design can already be used for creating fashion models that have enough realism for reproducing accurately the behavior of real garments. More than draping, our system is able to compute realistic animations thanks to good time-accurate numerical techniques applied to viscoelastic mechanical models of cloth.

The core technologies of this system are now being adapted to actual needs of the garment industry through collaborative projects, which deal on mechanical characterization of fabrics, virtual prototyping, manufacturing processes, e-commerce. Although some advances are still welcome in the area of efficiency and accuracy of mechanical simulation techniques, the challenge is now to create new tools that will ensure to the garment industry a smooth transition from tradition to novel possibilities offered by virtual simulation.



Figure 14: Virtual prototyping: Displaying weft constraints on an animated body (from standing to sitting). Element size is 5mm.

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